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Publisher's version / Version de l'éditeur:

<https://doi.org/10.4224/8895149>

Laboratory Memorandum; no. LM-2004-09, 2004

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DOCUMENTATION PAGE

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|--|--------------------------|----------------------------------|-------------------|
| REPORT NUMBER LM-2004-09 | NRC REPORT NUMBER | DATE 23 – April - 2004 | |
| REPORT SECURITY CLASSIFICATION Unclassified | | DISTRIBUTION | |
| TITLE Design of a Yaw Restraining Heave Post | | | |
| AUTHOR(S) Robert Shandera | | | |
| CORPORATE AUTHOR(S)/PERFORMING AGENCY(S) | | | |
| PUBLICATION | | | |
| SPONSORING AGENCY(S) | | | |
| IMD PROJECT NUMBER 421016 | | NRC FILE NUMBER | |
| KEY WORDS | | PAGES 16 | FIGS. 6 |
| TABLES | | | |
| SUMMARY <p>The Institute for Ocean Technology is a national research facility that specializes in the testing of scale models. In many cases they are models of vessels that may one day see production. The information gathered from the testing facilities will give the people who designed the vessel a sense of how the vessel will perform in the real world. Often the models are not self propelled, yet propulsion must come from the carriages above the tow tanks. Often the propulsion is transferred to the models through the use of a heave post. The current heave post does not have the ability to restrict the motion of the model about its own axis. Therefore another piece of equipment is used to accomplish this task. There are a number of problems associated with this device that are considered undesirable. To eliminate the use of this piece of equipment a new heave post was designed. In order to have a heave post which functioned properly a set of design criteria were established. The design criteria lead to the development of several designs, from which the best suited candidate was chosen. The new heave post, when it is manufactured will supplement the existing heave post, giving the facility the ability to operate more smoothly.</p> | | | |
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océaniques

Design of a Yaw Restraining Heave Post

LM-2004-09

Robert Shandera

April 2004

Summary

The Institute for Ocean Technology is a national research facility that specializes in the testing of scale models. In many cases they are models of ships and structures that may one day see production. The information gathered from the testing facilities will give the people who designed the vessel a sense of how the vessel will perform in the real world. Often the models are not self propelled, yet propulsion must come from the carriages above the tow tanks. Often the propulsion is transferred to the models through the use of a heave post. The current heave post does not have the ability to restrict the motion of the model in yaw. Therefore, the grasshopper is used to accomplish this task. There are a number of problems associated with this device that are considered undesirable. To eliminate the use of this piece of equipment a new heave post was designed. In order to have a heave post which functioned properly a set of design criteria were established. The design criteria lead to the development of several designs from which the best suited candidate was chosen. The new heave post, will supplement the existing heave post giving the facility the ability to operate more smoothly since testing will occur in both tanks simultaneously.

Table of Contents

| | |
|---|-----------|
| SUMMARY | I |
| TABLE OF FIGURES | II |
| 1.0 INTRODUCTION..... | 1 |
| 2.0 EXISTING DESIGN | 2 |
| 3.0 DESIGN CRITERIA | 6 |
| 4.0 DESIGN | 7 |
| 4.1 ALTERNATIVE HEAVE POST DESIGNS | 8 |
| 4.2 CHOSEN DESIGN..... | 9 |
| 4.3 FRAME DESIGN | 10 |
| 5.0 FUTURE CONSIDERATIONS | 15 |
| 6.0 CONCLUSION..... | 15 |
| 7.0 APPENDIX – A: FABRICATION DRAWINGS | |
| 8.0 APPENDIX – B: CALCULATIONS | |

Table of Figures

| | | |
|------------------|---|-----------|
| Figure 1. | The existing medium heave post..... | 2 |
| Figure 2. | The current yaw restricting "The Grasshopper". | 4 |
| Figure 3. | Large tow post..... | 7 |
| Figure 4. | The design drawing for the yaw restraining heave post is shown above. | 9 |
| Figure 5. | An early design of tow post support system. | 11 |
| Figure 6. | The final design of the frame..... | 12 |
| Figure 7. | Final assembled design..... | 15 |

1.0 Introduction

The Institute for Ocean Technology (IOT) is a part of the National Research Council (NRC). The main component of their research is model testing. The models that are tested at IOT's facilities consist of virtually any structure that will interact with the ocean. Everything from the Hibernia platform to underwater vehicles has been tested in the facilities at IOT. A majority of the models tested at IOT are scale models of ships. These models are tested to determine their performance under various conditions. In most of these tests, models require some form of propulsion. Some models are self-propelled. However, more are propelled through the carriages situated over the various tanks. The clear water towing tank (CWTT) carriage is capable of reaching a top speed of 10m/s. The device employed to provide propulsion varies depending on the type of model being tested, many times a simple heave post is used. This post is designed to allow the model to move freely in the vertical direction (heave), and rotate freely about that axis of the post (yaw) and is used in conjunction with a gimble that allows the model to pitch and roll. However it is not always desirable to allow the model to yaw. If a large model is unstable it is possible for it to yaw until it breaks the gimble. The gimble has the ability to resist yaw moments but has no strength in that direction. Currently IOT does have the ability to keep a model from yawing. However, this will be discussed later. This was the driving force for the need of a yaw restraining heave post. This post is not a complete replacement of the existing heave post. It will instead be used to supplement the

existing post. This will allow both the ice tank and the fresh water tow tank to operate at the same time without equipment availability causing a problem.

2.0 Existing Design

There are several tow posts at IOT. They vary in both diameter and length. Currently the medium heave post and gimble are used more often for the purposes of experimentation. The dimensions of that post are:

- Outer Diameter 2.5"
- Inner Diameter 1.75"
- Weight 22.5 lbs
- Length 23"



Figure 1. The existing medium heave post

This post can be seen in figure 1. Also shown in this figure is the frame that supports the post. The post is allowed to move freely in both heave and yaw. It

is not allowed to move in the horizontal direction. This is obviously a necessity if the post is to transfer any force to the model. The heave post is strong enough to resist virtually any force applied via a model. The limiting factor of this design is the linear bearings that restrain the yaw of the model. The average model is accelerated at 0.4m/s^2 , and the operator of the carriage gradually accelerates large models to prevent the bearings from becoming overloaded. However the real danger of failure occurs when the operator must fully apply the brakes of the carriage. This results in an deceleration of 3 m/s^2 . Forces much greater in magnitude are generated at this acceleration. A quick inspection of Newton's second law shows that the same model will generate a force 7 times larger occurs during a sudden stop than under normal operating conditions.

$$F = ma \quad (1)$$

Where: F = force
 m = mass
 a = acceleration

The medium heave post does perform the job it was designed to do. However the need to restrict the motion of the model about the vertical axis presented itself.

The institute is currently able to restrict the motion of the model in yaw. This is done through application of a device known within the institute as "the grasshopper". The grasshopper can be seen in figure 2. The grasshopper

attaches to the bow of the model and the square post is used to resist the yaw. There are two major problems associated with the grasshopper. First, there is a counterweight that can be seen to the left of the figure 2. When the model has trimmed, the counterweight has no effect. However, when a model is pitching and heaving due to wave and wind interactions, the counterweight has a detrimental effect. When wave and wind are applied to a model the intention is to measure how the model responds.



Figure 2. The current yaw restricting "The Grasshopper".

The counterweight does not allow the true motion of the model to be observed. The counterweight actually has a dampening effect on the motion of the model. This reduces the accuracy of the data collected and does not allow for a good correlation between model motion and full scale motion. The second problem associated with the grasshopper is its length. The grasshopper is not always

long enough to reach the model and hold it at the appropriate angle. When this situation occurs a number of six-inch extensions are added. This reduces the flexural and torsional stiffness of the apparatus. The grasshopper acts like a cantilever beam. As the length of the cantilever beam increases, so does the amount the end of the beam will deflect under a given load. Also, as the length increases, so does the angle of twist that will be experienced. These phenomena are determined by the following equations:

$$\theta = \frac{TL}{GJ} \quad (2)$$

Where:

- θ = the angle of twist
- T = the applied torque
- L = the length of the member
- G = the modulus of rigidity
- J = the polar moment of inertia of the cross section

$$\delta = \frac{PL^3}{3EI} \quad (3)$$

Where:

- δ = the deflection of the end of the post
- P = the applied load
- L = the length of the post
- E = the modulus of elasticity
- I = the moment of inertia

3.0 Design Criteria

The design of the tow post is a classic design problem. The post is required to be longer, lighter, and stronger than the current post. This does not give any specific limit to the characteristics of the design. It was determined that the post must meet the following characteristics:

- Weight: 45Lbs.
- Heave: 27in. (amount of heave for long medium diameter post.)
- Model Size:
 - Length 7m
 - Mass 1000 kg
- Acceleration: 3m/s^2
- Post must mount to the large tow post shown in figure 3.
- The ability to resist yaw forces while being able to trim the yaw angle.
- The ability to lock the post in heave must be present.



Figure 3. Large tow post.

4.0 Design

There were a number of possible designs considered for the new heave post. There are several characteristics that are common among all the designs. First, the inclusion of a linear rail and bearing system is common to all designs. The linear rail and bearing insure that the post will heave properly. That is, the coefficient of friction will be lowered greatly, which will keep the heave post heaving freely. Also, the bearings will be used to counteract any yaw force acting on the model. The bearings chosen for the new heave post are THK SHS 15LC. These rails were chosen over the THK SHS 15C bearings because they have a higher critical load of 35kN. Under an applied load of 35kN the balls in the bearing will start to cause indentations on the rails that will decrease the

effectiveness of the rail system. Also, all the designs must be mounted in some way to the front of the existing tow post shown in figure 3. This will allow the cross-sectional area of the heave post to be increased in comparison to the existing design. An increase in cross-sectional area will increase the strength of the post in virtually all-loading conditions. This is important since the length of the post is also going to increase. The fact that the heave post is to be mounted away from the center of the tow post will allow for easy adjustment of the yaw angle of the model with respect to the tank wall. It is possible to simply have a pivot post with a treaded rod at another location to adjust the yaw angle with the heave post in this location. However, if the post were aligned with the center of the tow post, a much more complicated gearing system would likely be used.

4.1 Alternative Heave Post Designs

There were a number of alternative designs considered for the heave post. They consisted of different geometries and the use of different materials. The first design considered was simply an alteration of the current heave post. A 2.5" outer diameter hollow shaft was considered. The existing design is able to withstand any force applied to the model. Again the linear bearings are the limiting factor to the force that can be applied. This design would difficult to fabricate. The post would need to be machined to allow for proper alignment of the linear rails. Even though it is possible for this to be done with the boring mill it would require more work and reduce the strength of the post. Then, a carbon fiber square post with external dimensions of 3" x 3" was tried. The use of a

carbon fiber post would have reduced the weight of the post and will not be sacrificing a great deal of strength. However, the technology is not present at IOT to manufacture such a post, purchasing a carbon fiber sail boom was considered. A closer inspection was made to the process that will be used to attach the linear rails. The process would be very complicated and greatly increase the price of the design. If this technology were more readily available at IOT then this material would have become a more viable option.

4.2 Chosen Design

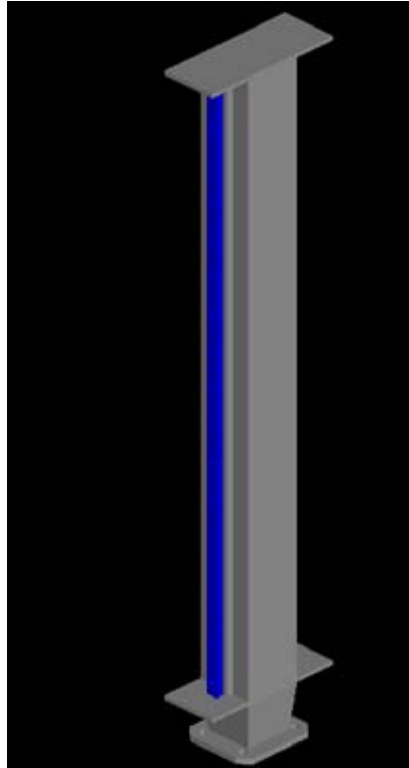


Figure 4. The design drawing for the yaw restraining heave post is shown above.

The chosen design is that of a 3" x 3" x 3/16" steel box tube with a length of 40".

The use of the box tube not only increased the moment of inertia but also

reduced the weight of the post. However, the addition of the linear rails and the mounting surfaces for those rails increased the weight to approximately 40lbs. The new design is heavier than the existing design, however for this is to be expected considering the new design is 17" longer than the current design. The weights of 40" length post are 23lb and 35lbs for the new and existing design respectively. This is before the addition of the linear rail. The maximum force that will be experienced when the post is in operation is determined by the maximum acceleration of the carriage. Under normal operating conditions an acceleration of 0.4m/s^2 will be experienced. However, it is possible that an acceleration of 3m/s^2 will be experienced when the carriages brakes are fully applied in an emergency. The post may be able to withstand large forces, but without a support system it is not able to propel a model.

4.3 Frame Design

The support system for the tow post underwent the most drastic changes from the beginning of the design process. The design of the frame of the post was an evolutionary process. When a new feature was identified the design of the support system changed. The major functions that the frame required to possess are:

- Allow the post to heave freely.
- Resist yaw forces.
- Allow for the attachment of the existing wake flow survey equipment.
- Have the ability adjust the model's yaw angle
- Have the ability to lock the post in heave

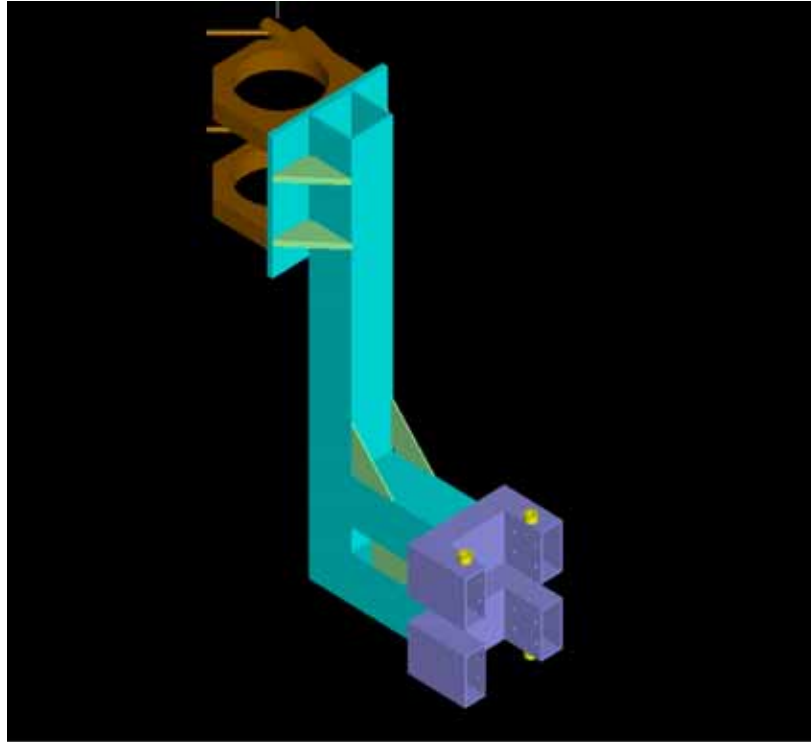


Figure 5. An early design of tow post support system.

The design for the frame started very simply. An early design for the post frame can be seen in figure 5. This design was very simple and very light. It is simply a number of pieces of 4" x 4" box tube welded together. This design was very strong, but there are a number of problems associated with the design. First, there is no way to preload the linear bearings. This is a technical problem that will affect the effectiveness of the device. During assembly the bearings position must be adjusted and pre-loaded until the post heaves freely. This is, of course, a major problem that has been eliminated in future iterations of the heave post frame. Another problem associated with this design is the spacing between the bearings. It is common practice to provide at least one bearing length between the bearings. This helps to reduce the amount of force acting on each bearing.

The frame is much too long, and will behave like a cantilever beam. Like all cantilever beams as the length increases the end of the post will deflect much more. Also this frame does not allow the yaw angle to be trimmed. This is a necessity to eliminate any error introduced during the installation of the frame.

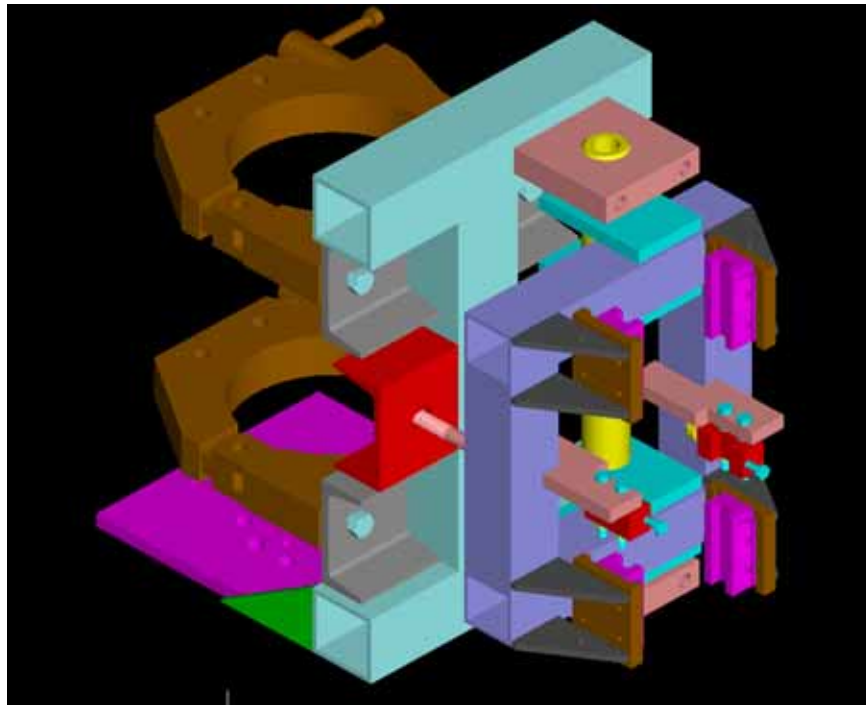


Figure 6. The final design of the frame.

The final design of the heave post frame is shown in figure 6. This design has incorporated all function that was necessary. This increased the number of components needed to full fill the design criteria. As the number of components increased so did the weight. However, for the components possessing the largest volume, the large tow post clamps are to be made of aluminum to reduce their weight. The weight is not a major problem as far as testing is concerned. The tow post will support the weight of the frame. The only weight that is of major concern is that of the heave post. The design of the frame is very

compact. It is not possible to see the scale of this design from figure 6.

However, the frame is approximately 23" high and 12" wide. In fact, it will be possible to restrict the motion of heave post and crane the heave post into position while it is attached to the large tow post. This makes the job of the people setting up the models in the testing facilities easier. They will no longer need to move the post into position with a boat and have the risk of falling into the tow tank. There are a number of features that this frame possesses that the first design does not. It is now possible to adjust the yaw angle of the model. This will insure that the model will run straight down the tank. However, the amount of adjustment present is only one or two degrees in either direction. This all that is required to trim any error introduced due to the installation. Another option that needed to be designed is a system to restrain the post in heave. In the case of a wake flow survey the model is set at a certain level in the water and a certain pitch angle is also imposed. The axial loading on the post during claming must not be applied to the bearings. If either the weight of the model or the buoyant force is too great, then the bearings may fail. Therefore, it was decided to attach the heave restraint to the frame and clamp to the linear rails. The final frame design was also required to have one final feature that needed to be added. This was the ability to impose an angle on the model and it is also a necessity. There is equipment existing already for this purpose. After a number of potential designs it was determined that attaching the existing equipment would be simpler and more cost effective. This required a slight modification to the large clamps to accommodate two bolt holes. This will allow both tanks to

conduct testing at the same time without equipment being an issue. The new design of both the heave post and its corresponding frame are more complicated than the existing heave post. However, more options are available and the major problem associated with the grasshopper has been eliminated. Even though a number of improvements have been made over the existing post, the new post may need some improvements in the future. The final design can be seen in figure 7.

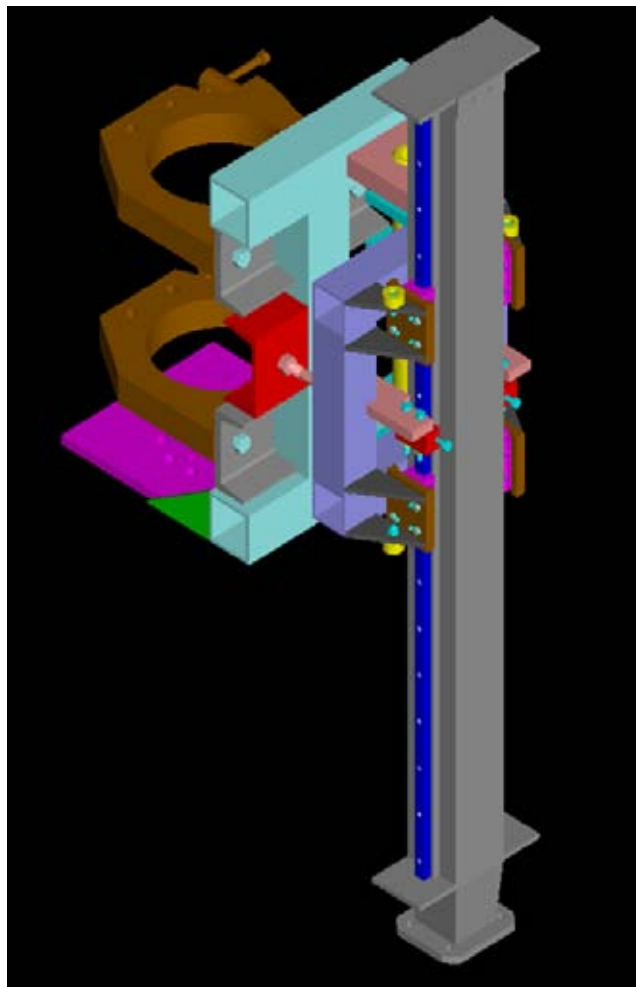


Figure 7. Final assembled design.

5.0 Future Considerations

There are a number of areas in which the heave post and its frame can be improved. As the frame is currently designed it is impossible to measure the amount of yaw force present in the model. This is due to the positioning of the pivot post in relation to the tow post. The pivot post is not in alignment with the tow post. This means that the treaded rod will carry a certain amount of the tow force. Since a portion of the tow force will always be present there will never be a time when there is no force acting on the bolt. This makes the ability to tune the yaw almost impossible. In the future this process can be automated. One possibility could be to use a ball screw that is able to pivot, and a stepper motor. A motor controller will adjust the angle of the post slightly by spinning in a given direction. Then a micro controller will compare the force experienced by the load cell to a previous measurement. If the force is lower then the screw will advance another step. However, if the measured force is greater then the previous it will reverse direction. This will continue to happen until the model is oriented in the proper position.

6.0 Conclusion

The design of the new heave post has been a success up to this point. The design passed the final design review and was approved for fabrication. The fabrication drawings can be seen in appendix A. The design exceeded the existing design in all areas. The only factor that is keeping this design on the

shelf will be funding. When funding is available this post will be fabricated and quickly put into use. This post will be a great asset to the Institute for Ocean Technology because it will help get more accurate readings on model runs. Also, the fact that IOT will now have two towing kits should decrease the likelihood that scheduling problems between the two tow tanks will be reduced. This piece of equipment will be an invaluable addition to IOT's current method of propelling models.

7.0 Appendix A

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| Drawing Number | Title | Page Size |
|----------------|-------------------------|-----------|
| 421016100 | Drawing List | A3 |
| 421016101 | Parts List | A3 |
| 421016102 | Total Assembly | A3 |
| 421016103 | Sub Assembly 1 | A2 |
| 421016104 | Sub Assembly 2 | A2 |
| 421016105 | Sub Assembly 3 | A3 |
| 421016106 | Front Frame Parts | A3 |
| 421016107 | Shaft Supports | A3 |
| 421016108 | Pivot Shaft | A3 |
| 421016109 | Pitch Adapter Parts | A2 |
| 421016110 | Pitch Apoter assembly | A3 |
| 421016111 | Large Clamps | A3 |
| 421016112 | Clamp Parts | A3 |
| 421016113 | Clamp Handle | A3 |
| 421016114 | Heave Clamp Part | A3 |
| 421016115 | Heave Clamp Parts | A3 |
| 421016116 | Back Frame Parts | A3 |
| 421016117 | Back Frame Parts 2 | A3 |
| 421016118 | Bearing Mounts | A3 |
| 421016119 | Front Frame Fabrication | A3 |
| 421016120 | Bearing Mount Machining | A3 |
| 421016121 | Post Pieces | A3 |
| 421016122 | Toper Pieces | A3 |
| 421016123 | Toper Fabrication | A3 |
| 421016124 | Bearing Mounting plates | A3 |
| 421016125 | Post Fabrication | A3 |
| 421016125 | Post Machining | A3 |

Notes:

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421016

Yaw Restraining Tow Post
Drawing List

TOLERANCES
(unless specified)

0.X ± 0.03
0.XX ± 0.015
0.XXX ± 0.005

Angle +/- .5 deg.
Fabrication +/- .04
Fraction (5 inch 1/4-1/2)

Material
44W Steel
Heat treatment

FINISH

DIMENSIONS IN INCHES
☒ DECIMALS ☐ FRACTIONS

UNIT

DATE
4/10/16

DRAWN
R. Shandera

APPROVED

Quantity
NA

SCALE
1 : 10

DATE
19-Feb-1999

PAGE
1 of 1



List of Materials

| # | Description | Size | Quantity |
|----|--------------------------------|---------------|----------|
| 1 | 4"x4"x3/16" steel box tube | 40" long | 1 |
| 2 | 2"x2"x3/16" steel Box Tube | 10.28" long | 2 |
| | | 8" long | 2 |
| 3 | 2.5"x2.5"x3/16" Steel Box Tube | 13.5" long | 1 |
| | | 10.1" Long | 2 |
| 4 | 2" Steel Plate | 4"x4.5" | 1 |
| 5 | 1/2" Steel Plate | 9.5" x 9.5" | 1 |
| | | 2.75" x 2.75" | 2 |
| | | 2.75" x 1.5" | 2 |
| | | 1.75" x 4" | 3 |
| 6 | 1/4" Steel Plate | 3" x 2.5" | 16 |
| | | 2.5" x 4" | 2 |
| | | 2.25" x 3" | 2 |
| 7 | 3/16" Steel Plate | 3" x 3" | 4 |
| 8 | 1" Steel Plate | 2" x 4" | 4 |
| 9 | [4 x 7.25 Steel Channel | 3.375" Long | 2 |
| | | 3" Long | 1 |
| 10 | 1.5" Alum plate | 10.5" x 6" | 1 |
| | | 4.5" x 9.5" | 1 |
| 11 | 1/2" dia steel rod | 4" long | 1 |
| 12 | 1.5" dia rod | 5.5" long | 1 |
| | | 3" long | 1 |
| 13 | 4" x 4" X1/4" steel box tube | 20" long | 1 |
| 14 | 2" dia Steel Shaft | 16" long | 1 |

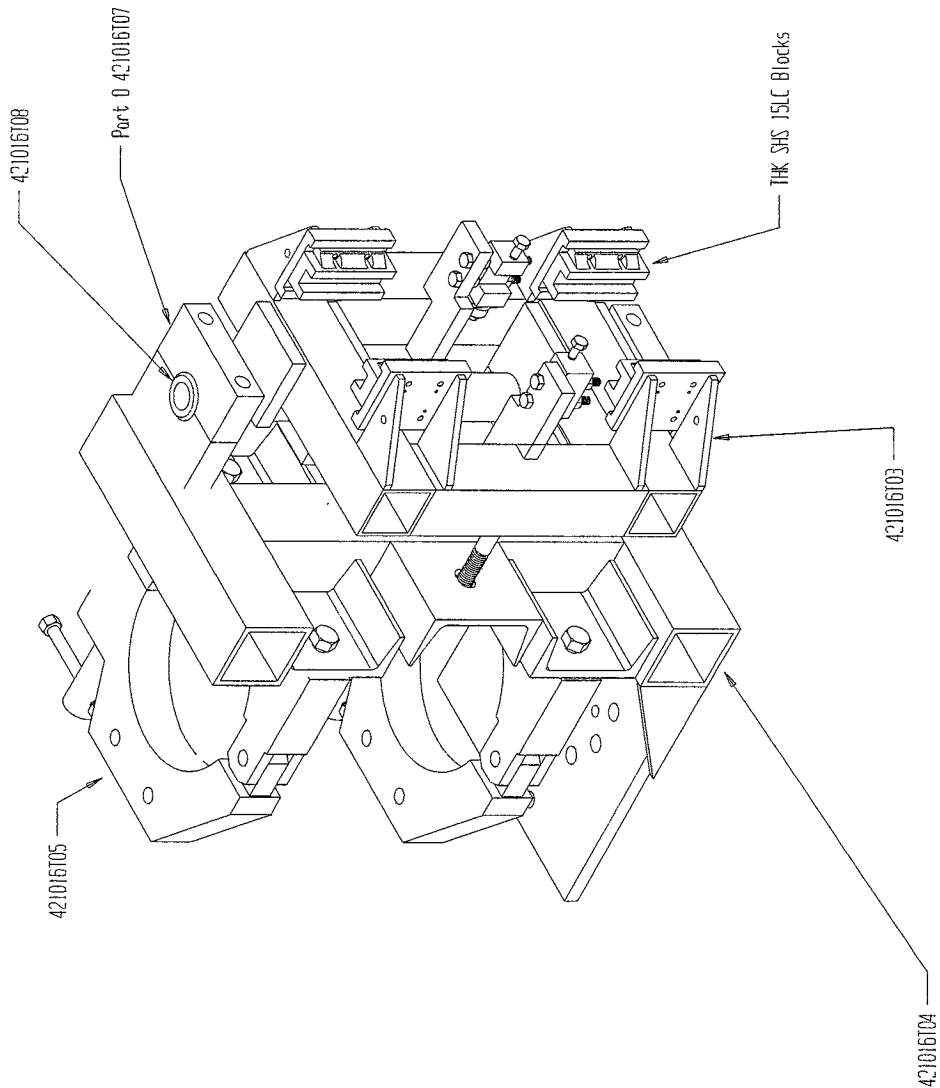
List of Materials (cont'd)

| # | Description | Size | Quantity |
|----|-------------------|------------|----------|
| 15 | 3/8" shaft | 4" long | 1 |
| 16 | THK SHS 15L Black | | 4 |
| 17 | THK SHS 15 Rail | 39.5" long | 2 |

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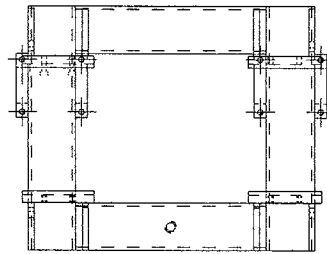
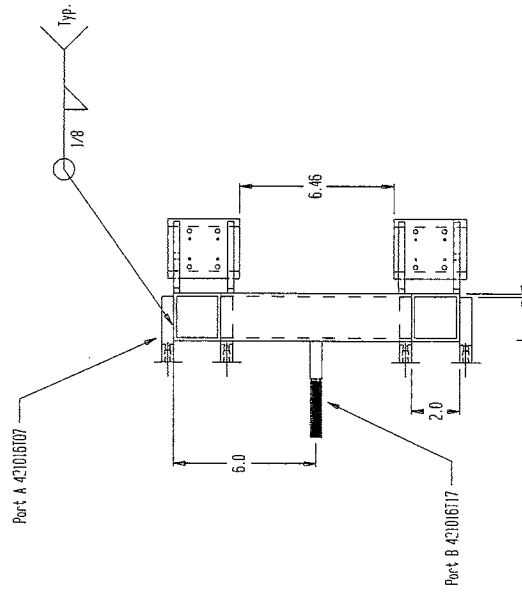
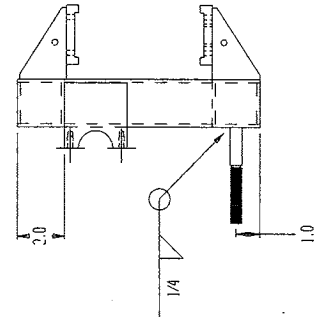
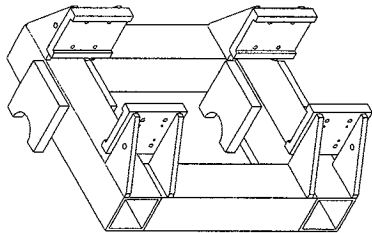
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|--|--|--|---|
|  National Research Council of Canada Conseil national de recherches Canada | | ARC-CNR Institute for Ocean Technology Perwin Place, P.O. Box 12093, Postal Station A St. John's, Newfoundland A1B 3T5 | |
| TOLERANCES (unless specified) 0.X ± 0.03 0.XX ± 0.015 0.XXX ± 0.005 Angle +/- .5 deg. Fabrication +/- .04 Fraction < 6 inch +/- .004 > 6 inch +/- .020 | Material 44W Steel Heat treatment | ITEM 421016 QTY 1 APPROVED R. Shandera DATE 31-Apr-11-2004 | ITEM 421016 QTY 1 APPROVED R. Shandera DATE 31-Apr-11-2004 |
| FUNCTIONS IN: <input checked="" type="checkbox"/> JACKS <input type="checkbox"/> ALL INTERIORS  | | YAW RESTRAINING TOW POST LIST OF MATERIALS 421016 A3 421016101 | |

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Notes:
 Deburr - Remove All Sharp Edges
 Part File 421016T02

| | | | |
|---|---|---------------------------------------|---------------------|
| National Research Council of Canada Conseil national de recherches Canada | | | |
| Institute for Ocean Technology Perwin Place, P.O. Box 12093, Postal Station A St. John's, Newfoundland A1B 3T5 | | Yaw Restraining Tow Post Assembly | |
| Material 44W Steel Heat Treatment | FINISH DIMENSIONS IN INCHES DIMENSIONS IN MILLIMETERS | DRAWN R. Shandera | CHECKED A3 |
| TOLERANCES (unless specified) 0.X ± 0.03 0.XX ± 0.015 0.XXX ± 0.005 Angle +/- .5 deg. Fabrication +/- .04 Fraction < 8/64 1/16 > 8/64 1/8 | QUANTITY 1 | PART NAME Yaw Restraining Tow Post | DATE 19-Apr-2004 |



Notes:
Deburr - Remove All Sharp Edges
Chip and Brush all welds
Part File 421016103

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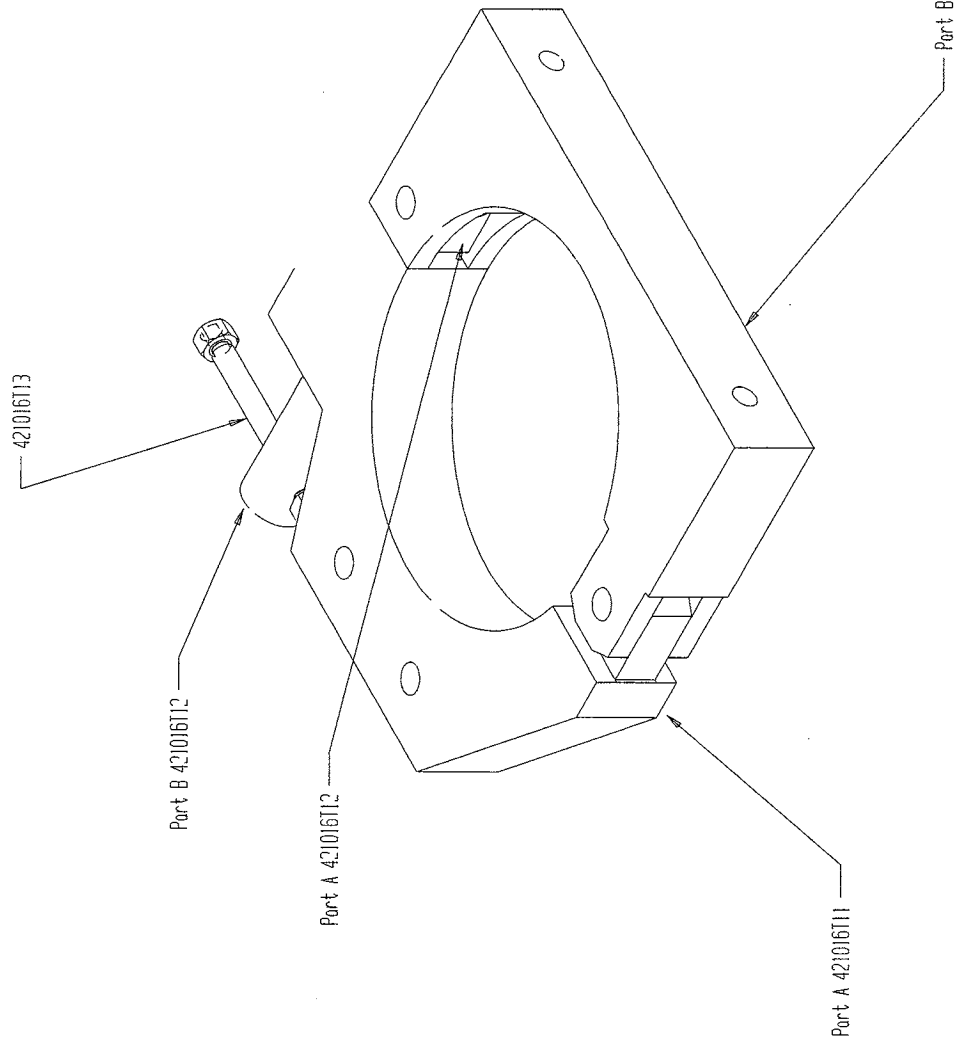
| | | | | |
|---|---|--|--|----------------------------|
| National Research Council Canada Conseil national de recherches Canada | Material HPS Heat Treatment FINISH CORROSION IN: <input checked="" type="checkbox"/> INKET <input type="checkbox"/> MILLING: <input type="checkbox"/> | Part 421016 Yaw Restraining tow post Sub Assembly | | DATE 14-FEB-1993 |
| | | Quantity NA | | DATE 14-FEB-1993 |
| TOLERANCES (Unless specified) 0.X ± 0.03 0.XX ± 0.015 0.XXX ± 0.005 Angle +/- .5 deg. Fabrication +/- .04 Fraction < 6 inch +/- 1/64 > 6 inch +/- 1/32 | | Company Institute for Ocean Technology Perwin Place, P.O. Box 12093, Postal Station A St. John's, Newfoundland A1B 3T5 | | |
| Part Name Yaw Restraining tow post Sub Assembly | | Part Number 421016103 | | |

ARC-CARC

| NO. | DATE | DESCRIPTION | REVISIONS | DATE | APPROVED |
|-----|------|-------------|-----------|------|----------|
| | | | | | |
| | | | | | |

Notes:

Deburr - Remove All Sharp Edges
Part File 421016T05



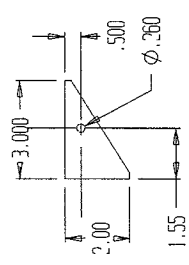
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|---|--|---|--|
| National Research Council Canada Conseil national de recherches Canada | | Institute for Ocean Technology Yermine Place, P.O. Box 12093, Postal Station A St. John's, Newfoundland A1B 3T5 | |
| TOLERANCES (unless specified) 0.X ± 0.03 0.XX ± 0.015 0.XXX ± 0.005 Angle +/- .5 deg. Fabrication +/- .04 Fraction < 6 inch 1/16 > 6 inch 1/32 | | TITLE 421016 Yaw Restraining Tow Post DRAWN R. Shandera APPROVED Quantity NO | |
| Material NO Heat treatment | | FINISH DIMENSIONS IN: INCHES FRACTIONS DECIMALS TYPED ANGLES | |
| PART NO 421016T05 | | DATE 19-Feb-1993 | |

| REV | DATE | DESCRIPTION | BY | APPROVED |
|-----|------|-------------|----|----------|
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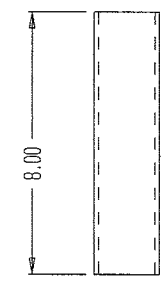
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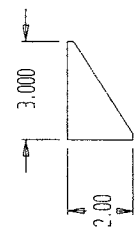
Part A
2 X 2 X 3/16 Box Tube
2 Needed



Part C
1/4 in Steel Plate
4 Needed



Part B
2 X 2 X 3/16 Box Tube
2 Needed



Part D
1/4 in Steel Plate
4 Needed

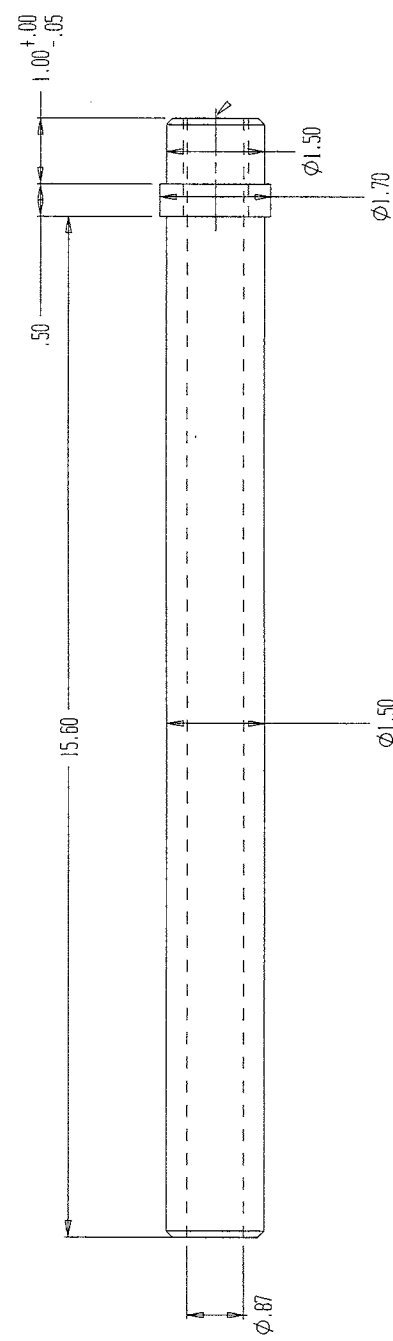
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|---|--|--|------------------|
| National Research Council Canada Conseil national de recherches Canada | | Institute for Ocean Technology Ferwin Place, P.O. Box 12093, Postal Station A St. John's, Newfoundland A1B 3T5 | |
| TOLERANCES (unless specified) 0.X ± 0.03 0.XX ± 0.015 0.XXX ± 0.005 Angle +/- .5 deg. Fabrication +/- .04 Fraction < 6 inch 1/4 1/32 > 6 inch 1/4 1/32 | Material 44W Steel Heat Treatment FINISH DIMENSIONS IN INCHES DECIMALS FRACTIONS ALL DIMENSIONS UNLESS OTHERWISE SPECIFIED | PART 421016 Yaw Restraining Tow Post Front Frame parts | REV 421016106 |
| DRAWN R. Shandera APPROVED Quantity 4 | | SCALE 1 : 4 DATE 08-April-2004 | |

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Notes:

Bechurr - Remove All Sharp Edges

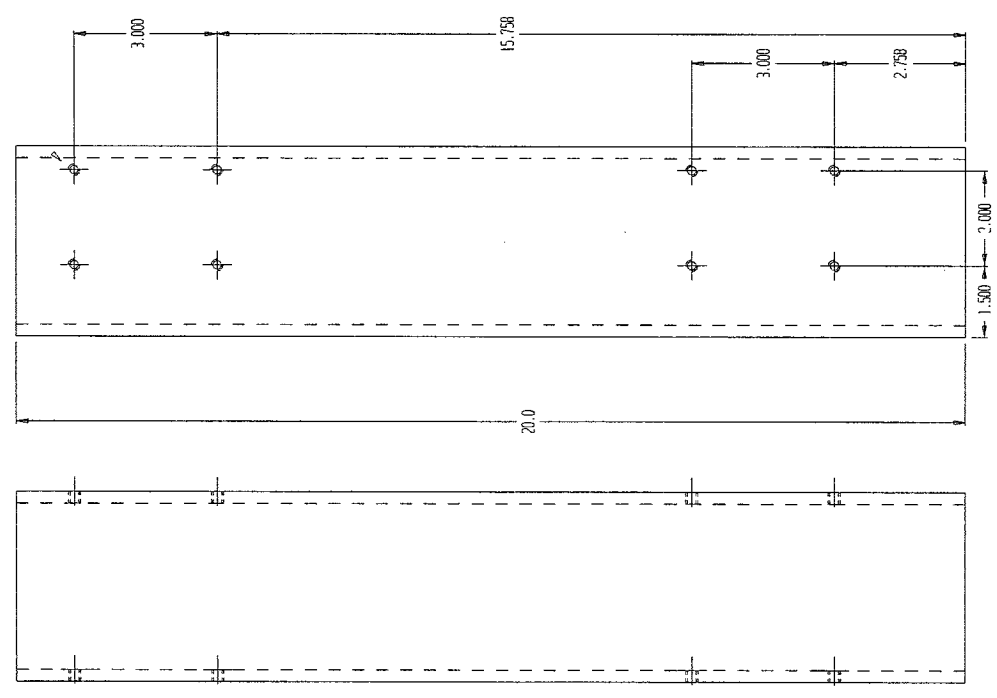
Part File 421016T08



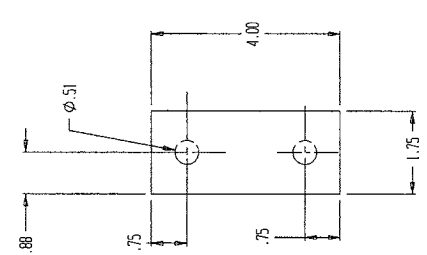
— 1-BUNC-28 ▽ 1

| | | | |
|---|--|--|--|
| | | National Research Council of Canada Conseil national de recherches Canada | |
| TOLERANCES (unless specified) 0.X ± 0.03 0.XX ± 0.015 0.XXX ± 0.005 Angle +/- .5 deg. Fabrication +/- .04 1/8 inch 1/4-1/2 | | Material 44W Steel Heat Treatment | |
| FINISH DIMENSIONING IN INCHES DECIMALS FRACTIONS | | INSTITUTE FOR DECAN TECHNOLOGY Perwin Place, P.O. Box 12093, Postal Station A St. John's, Newfoundland A1B 3T5 | |
| DATE 421016 | | TYPE Yaw Restraining Tow Post | |
| DRAWN R. Shandera | | CHECKED 421016T08 | |
| APPROVED Quantity 1 | | SCALE 1:1 | |
| DATE 6 April 2004 | | DRAWN 1 of 1 | |

16 1/4-20 UNC-2B



Part A
4' x 4' x 1/4' Steel Box Tube
1 Required

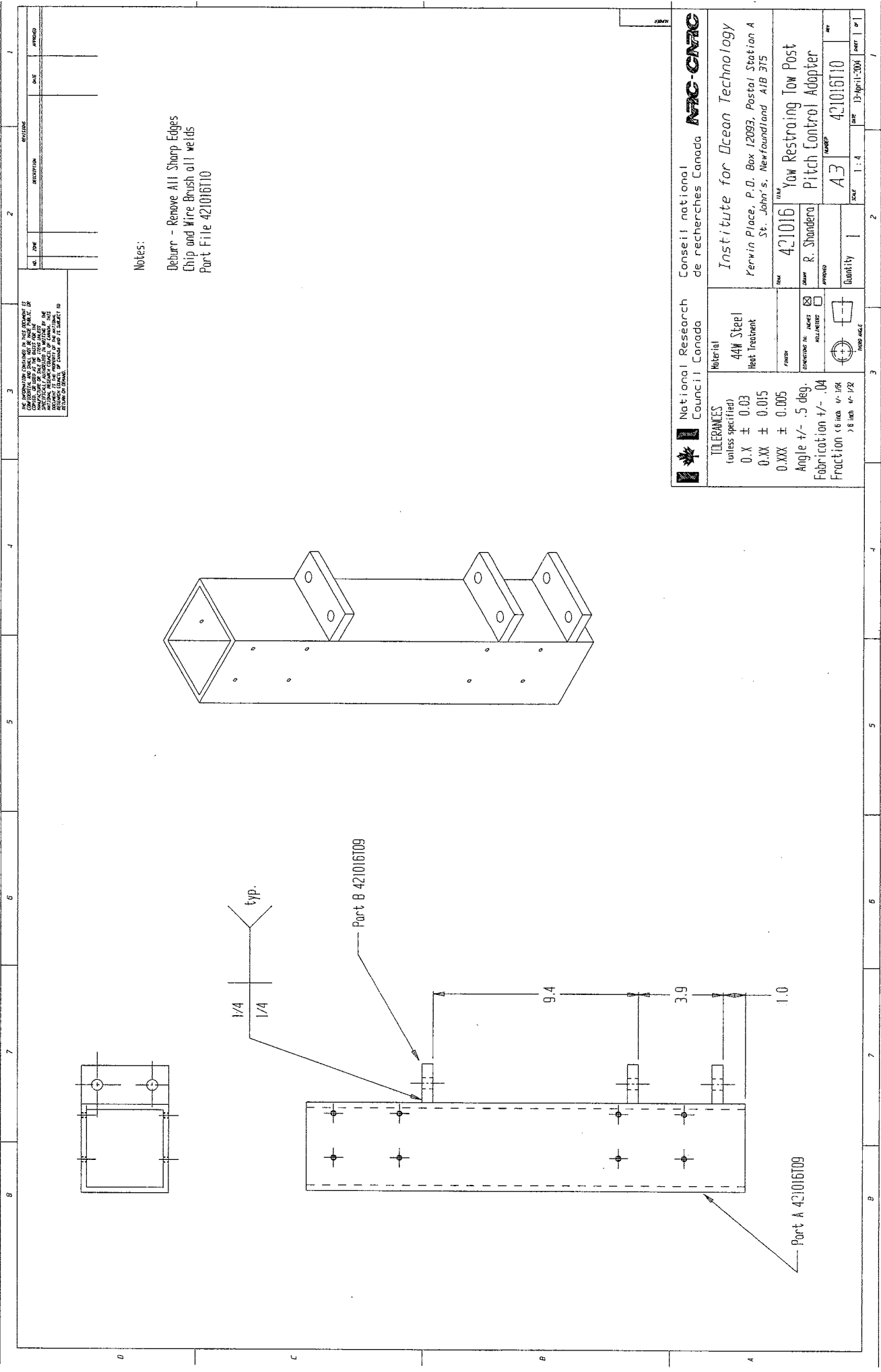


Part B
1/2' steel plate
3 Required

Notes:
Deburr - Remove All Sharp Edges
Part File 421016109

| REV | DATE | DESCRIPTION | APPROVED |
|-----|------|----------------|----------|
| 1 | | Initial Design | |
| 2 | | Revised Design | |
| 3 | | Final Design | |

| | | | | | |
|--|--|---------------------------------------|--|--|--|
| | | Conseil national de recherches Canada | | | |
| National Research Council Canada | | Institute for Ocean Technology | | Kerwin Place, P.O. Box 12093, Postal Station A St. John's, Newfoundland A1B 3T5 | |
| Material HSS Heat Treatment | | Part 421016 | | Title Yaw Restraining Tow Post Pitch Adapter Parts | |
| FINISH EMULSION IN OIL ALL SURFACES | | DRAWN R. Shandera | | DATE 08-April-2004 | |
| TOLERANCES (unless specified) 0.X ± 0.03 0.XX ± 0.015 0.XXX ± 0.005 Angle +/- .5 deg. Fabrication +/- .04 Fraction < 6 inch +/- .06 > 6 inch +/- .12 | | QUANTITY 95 noted | | SCALE 1:2 | |
| APPROVED | | PARTS A2 | | PARTS 421016109 | |



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Notes:

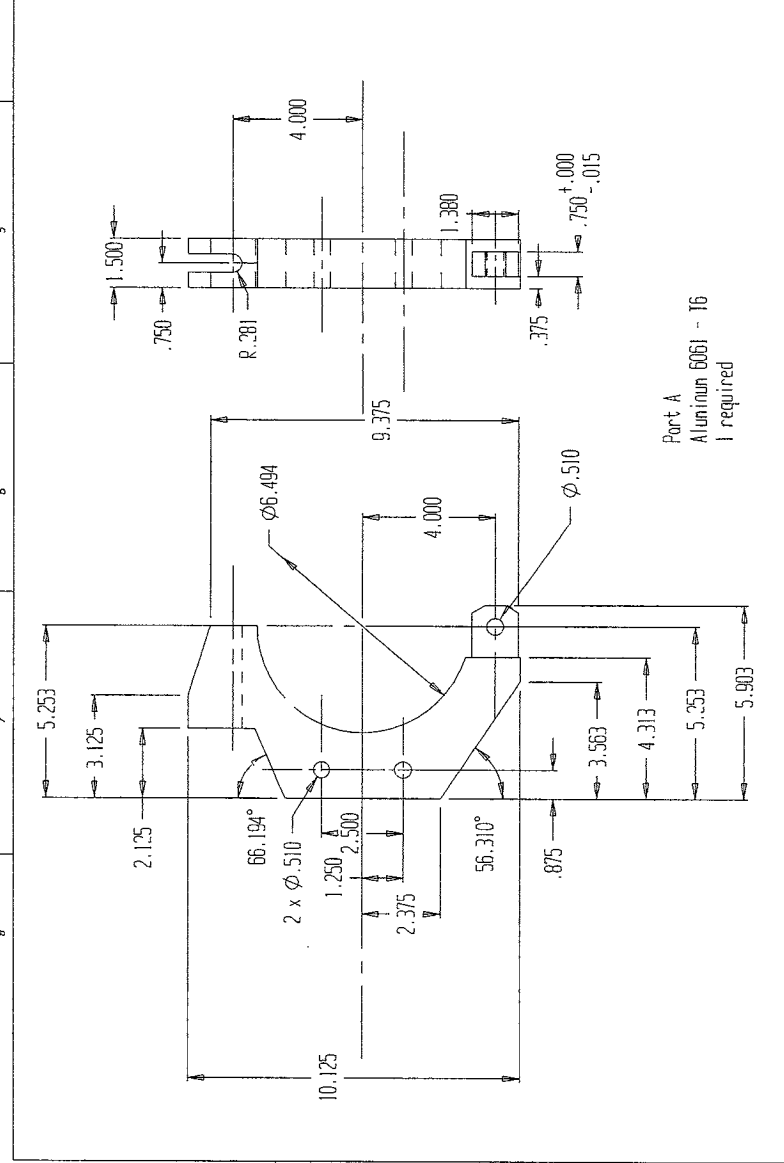
Debur - Remove All Sharp Edges
Chip and Wire Brush all welds
Part File 421016T10

| | | | |
|---|--|--|--|
| National Research Council of Canada Conseil national de recherches Canada | | Institute for Ocean Technology Yervin Place, P.O. Box 12093, Postal Station A St. John's, Newfoundland A1B 3T5 | |
| TOLERANCES (unless specified) 0.X ± 0.03 0.XX ± 0.015 0.XXX ± 0.005 Angle +/- .5 deg. Fabrication +/- .04 Fraction (6 inch +/- 1/64) (8 inch +/- 1/32) | | Material 44W Steel Heat Treatment | |
| FINISH DIMENSIONS IN INCHES <input checked="" type="checkbox"/> FULL FINISH <input type="checkbox"/> PARTIAL FINISH | | NAME 421016 TITLE Yaw Restraining Tow Post PITCH CONTROL ADAPTER | |
| DRAWN R. Shandera | | CHECKED A3 DATE 13-April-2004 | |
| QUANTITY 1 | | PART 421016T10 | |

| REV | DATE | DESCRIPTION | BY | APPROVED |
|-----|------|-------------|----|----------|
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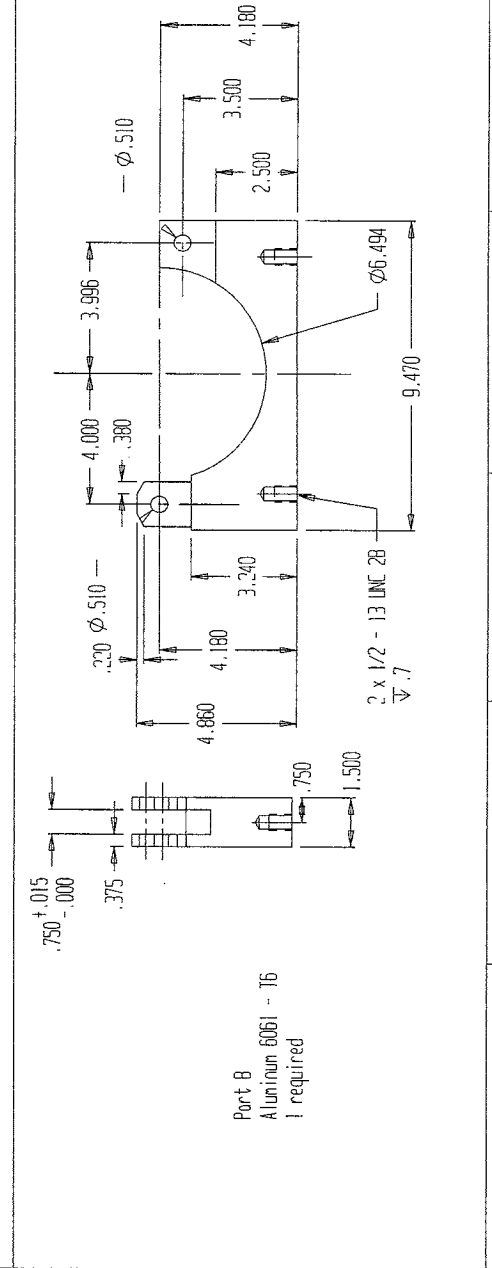
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Notes:
 Deburr - Remove All Sharp Edges
 Part File 421016T11



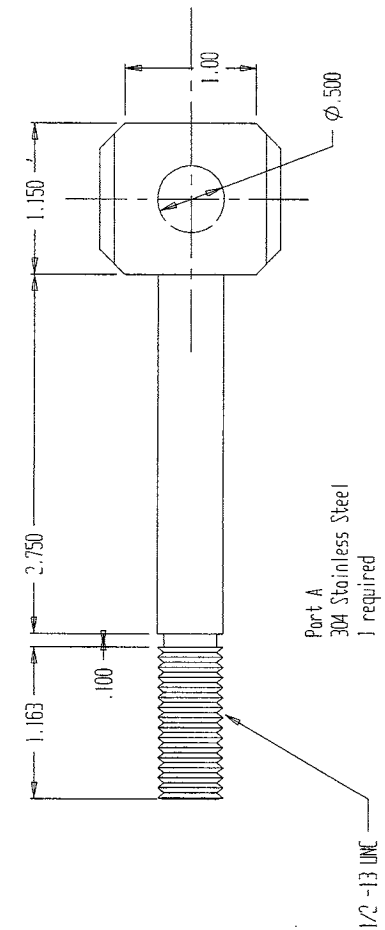
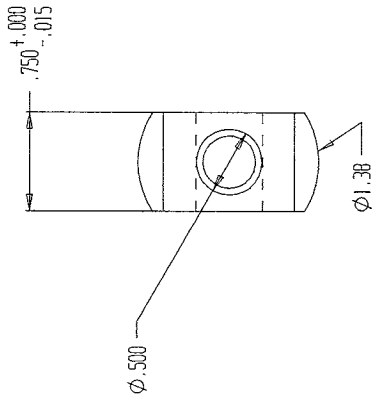
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| National Research Council Canada | | Conseil national de recherches Canada | | ARC-CARC | |
| Institute for Ocean Technology | | Institute for Ocean Technology | | Institute for Ocean Technology | |
| Herwin Place, P.O. Box 12093, Postal Station A | | Herwin Place, P.O. Box 12093, Postal Station A | | Herwin Place, P.O. Box 12093, Postal Station A | |
| St. John's, Newfoundland A1B 3T5 | | St. John's, Newfoundland A1B 3T5 | | St. John's, Newfoundland A1B 3T5 | |
| DATE | 421016 | DATE | 421016 | DATE | 421016 |
| DRAWN | R. Shandera | DRAWN | R. Shandera | DRAWN | R. Shandera |
| APPROVED | | APPROVED | | APPROVED | |
| Quantity | 1 | Quantity | 1 | Quantity | 1 |
| Part | 421016T11 | Part | 421016T11 | Part | 421016T11 |
| Rev | | Rev | | Rev | |

TOLERANCES (unless specified)
 0.X ± 0.03
 0.XX ± 0.015
 0.XXX ± 0.005
 Angle ± .5 deg.
 Fabrication ± .04
 Fraction < 6 inch 1/16
 > 6 inch 1/32

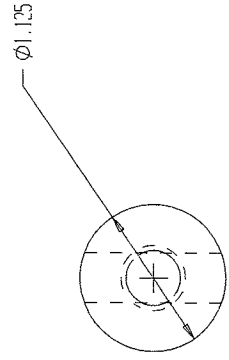
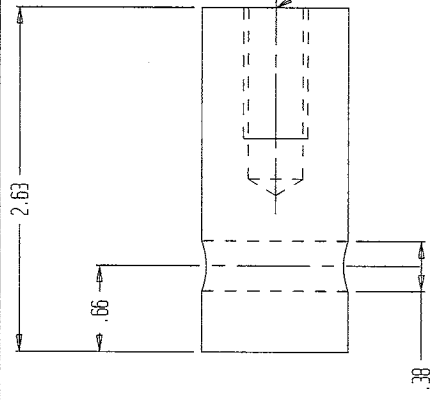


| REV | DATE | DESCRIPTION | BY | APP'D |
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Notes:
 Deburr - Remove All Sharp Edges
 Part File 421016T12



1/2 - 13 UNC

Ø 1.125

2.63

.38

Ø 500

Ø 1.38

.750 +.000 / -.015

1.163

2.750

1/2 - 13 UNC

Part A

304 Stainless Steel

1 required

Part B

304 Stainless Steel

1 Required

Ø 1.125

2.63

.38

Ø 500

Ø 1.38

.750 +.000 / -.015

1.163

2.750

1/2 - 13 UNC

Part A

304 Stainless Steel

1 required

Part B

304 Stainless Steel

1 Required

Ø 1.125

2.63

.38

Ø 500

Ø 1.38

.750 +.000 / -.015

1.163

2.750

1/2 - 13 UNC

Part A

304 Stainless Steel

1 required

Part B

304 Stainless Steel

1 Required

Ø 1.125

2.63

.38

Ø 500

Ø 1.38

.750 +.000 / -.015

1.163

2.750

1/2 - 13 UNC

Part A

304 Stainless Steel

1 required

Part B

304 Stainless Steel

1 Required

Ø 1.125

2.63

.38

Ø 500

Ø 1.38

.750 +.000 / -.015

1.163

2.750

1/2 - 13 UNC

Part A

304 Stainless Steel

1 required

Part B

304 Stainless Steel

1 Required

Ø 1.125

2.63

.38

Ø 500

Ø 1.38

.750 +.000 / -.015

1.163

2.750

1/2 - 13 UNC

Part A

304 Stainless Steel

1 required

Part B

304 Stainless Steel

1 Required

Ø 1.125

2.63

.38

Ø 500

Ø 1.38

.750 +.000 / -.015

1.163

2.750

1/2 - 13 UNC

Part A

304 Stainless Steel

1 required

Part B

304 Stainless Steel

1 Required

Ø 1.125

2.63

.38

Ø 500

Ø 1.38

.750 +.000 / -.015

1.163

2.750

1/2 - 13 UNC

Part A

304 Stainless Steel

1 required

Part B

304 Stainless Steel

1 Required

Ø 1.125

2.63

.38

Ø 500

Ø 1.38

.750 +.000 / -.015

1.163

2.750

1/2 - 13 UNC

Part A

304 Stainless Steel

1 required

Part B

304 Stainless Steel

1 Required

Ø 1.125

2.63

.38

Ø 500

Ø 1.38

.750 +.000 / -.015

1.163

2.750

1/2 - 13 UNC

Part A

304 Stainless Steel

1 required

Part B

304 Stainless Steel

1 Required

Ø 1.125

2.63

.38

Ø 500

Ø 1.38

.750 +.000 / -.015

1.163

2.750

1/2 - 13 UNC

Part A

304 Stainless Steel

1 required

Part B

304 Stainless Steel

1 Required

Ø 1.125

2.63

.38

Ø 500

Ø 1.38

.750 +.000 / -.015

1.163

2.750

1/2 - 13 UNC

Part A

304 Stainless Steel

1 required

Part B

304 Stainless Steel

1 Required

Ø 1.125

2.63

.38

Ø 500

Ø 1.38

.750 +.000 / -.015

1.163

2.750

1/2 - 13 UNC

Part A

304 Stainless Steel

1 required

Part B

304 Stainless Steel

1 Required

Ø 1.125

2.63

.38

Ø 500

Ø 1.38

.750 +.000 / -.015

1.163

2.750

1/2 - 13 UNC

Part A

304 Stainless Steel

1 required

Part B

304 Stainless Steel

1 Required

Ø 1.125

2.63

.38

Ø 500

Ø 1.38

.750 +.000 / -.015

1.163

2.750

1/2 - 13 UNC

Part A

304 Stainless Steel

1 required

Part B

304 Stainless Steel

1 Required

Ø 1.125

2.63

.38

Ø 500

Ø 1.38

.750 +.000 / -.015

1.163

2.750

1/2 - 13 UNC

Part A

304 Stainless Steel

1 required

Part B

304 Stainless Steel

1 Required

Ø 1.125

2.63

.38

Ø 500

Ø 1.38

.750 +.000 / -.015

1.163

2.750

1/2 - 13 UNC

Part A

304 Stainless Steel

1 required

Part B

304 Stainless Steel

1 Required

Ø 1.125

2.63

.38

Ø 500

Ø 1.38

.750 +.000 / -.015

1.163

2.750

1/2 - 13 UNC

Part A

304 Stainless Steel

1 required

Part B

304 Stainless Steel

1 Required

Ø 1.125

2.63

.38

Ø 500

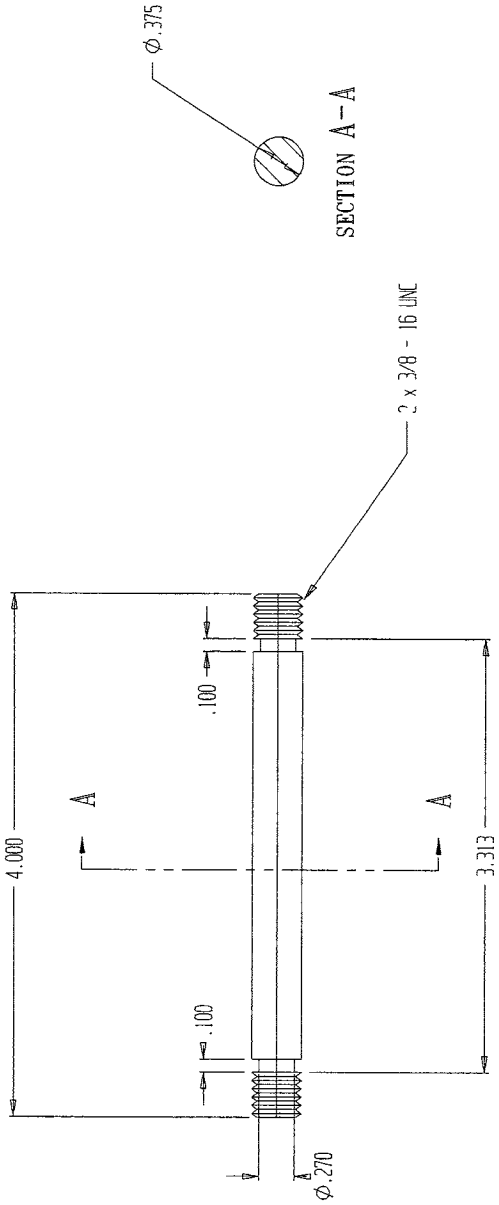
Ø 1.38

.750 +.000 / -.015

1.163

2.750

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SECTION A-A

Notes:

- Debur - Remove All Sharp Edges
- Part File 421016T13

| | | | |
|--|----------------------|--|------------------|
| National Research Council Canada Conseil national de recherches Canada | | NRC-CNRC | |
| Institute for Ocean Technology Terwin Place, P.O. Box 12093, Postal Station A St. John's, Newfoundland A1B 3T5 | | Yaw Restraining Tow Post Clamp Handle | |
| TITLE 421016 | DRAWN R. Shandera | CHECKED A3 | REV 421016T13 |
| QUANTITY 1 | SCALE 1:1 | DATE 13-Apr-13 | SHEET 1 OF 1 |
| TOLERANCES (unless specified) 0.X ± 0.03 0.XX ± 0.015 0.XXX ± 0.005 Angle +/- .5 deg. Fabrication +/- .04 Fraction (6 inch 4/16) > 6 inch 1/16 | | MATERIAL Steel Heat Treatment FINISH HANDLING IN JACKET POLLUTED YES <input checked="" type="checkbox"/> NO <input type="checkbox"/> | |

| | | | | | | | | | | | | | | | |
|----------|--|----------|--|----------|--|----------|--|----------|--|----------|--|----------|--|----------|--|
| 1 | | 2 | | 3 | | 4 | | 5 | | 6 | | 7 | | 8 | |
| DATE | | REVISION | | DATE | | DATE | | DATE | | DATE | | DATE | | DATE | |
| APPROVED | | APPROVED | | APPROVED | | APPROVED | | APPROVED | | APPROVED | | APPROVED | | APPROVED | |

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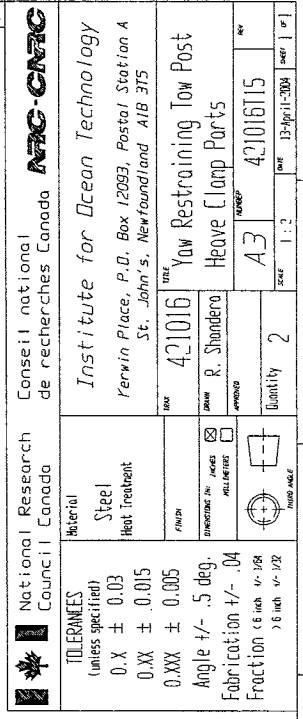
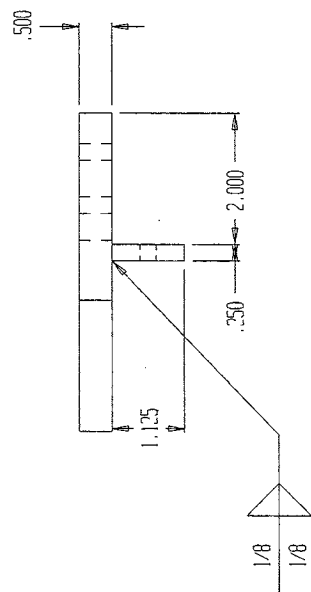
Notes:

Deburr - Remove All Sharp Edges

Part File: 421016114

| | | | |
|--|--|---|--|
| | | National Research Council Canada Conseil national de recherches Canada | |
| TOLERANCES (unless specified) 0.X ± 0.03 0.XX ± 0.015 0.XXX ± 0.005 Angle +/- .5 deg. Fabrication +/- .04 Fraction < 6 inch 1/16 > 6 inch 1/32 | | Material Steel Heat treatment | |
| FINISH DIMENSIONS IN: INCHES MILLIMETERS | | PART 878 R. Shoudera | |
| QUANTITY 2 | | PART 421016114 | |
| SCALE 1:1 | | DATE 13-Apr-2004 | |
| SHEET 1 OF 1 | | SHEET 1 OF 1 | |

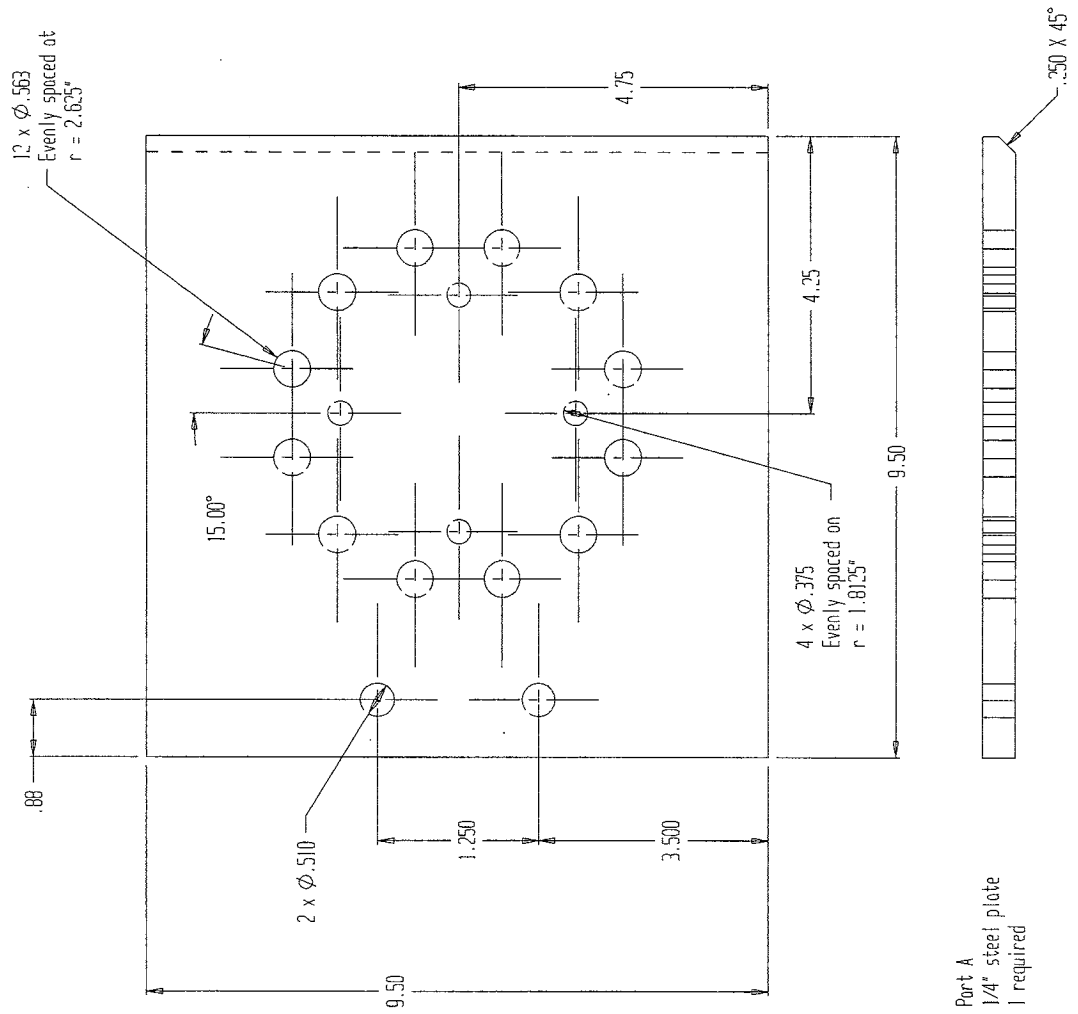
Conseil national de recherches Canada
 Institute for Ocean Technology
 Perwin Place, P.O. Box 12093, Postal Station A
 St. John's, Newfoundland A1B 3T5



[illegible]

| REV | DATE | DESCRIPTION | BY/CHK |
|-----|------|-------------|--------|
| | | | |
| | | | |

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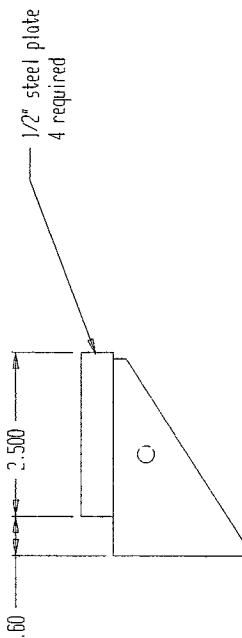


Notes:
Deburr - Remove All Sharp Edges
Part File 421016T17

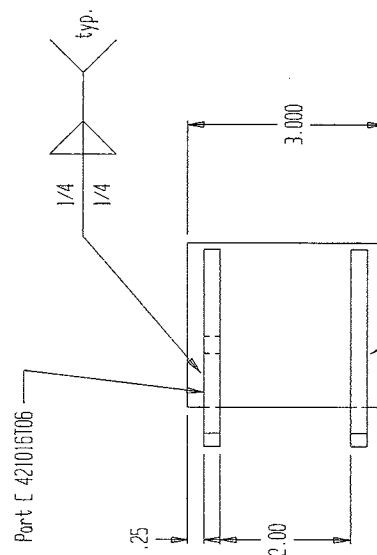
Part B
Steel
1 Required

Part A
1/4" steel plate
1 required

| | | | |
|---|--|--|--|
| | | National Research Council of Canada Conseil national de recherches Canada | |
| TOLERANCES (unless specified) 0.X \pm 0.03 0.XX \pm 0.015 0.XXX \pm 0.005 Angle \pm .5 deg. Fabrication \pm .04 Fraction \leq 6 inch \pm .001 $>$ 6 inch \pm .002 | | Material OS noted Heat treatment | |
| FINISH DIMENSIONS IN: <input checked="" type="checkbox"/> INCHES <input type="checkbox"/> MILLIMETERS DECIMALS: <input type="checkbox"/> <input checked="" type="checkbox"/> THIRDS | | PART NAME 421016 Yaw Restraining Tow Post Back from parts 2 | |
| QUANTITY OS noted | | DATE 13-Apr-2004 | |
| DRAWN BY R. Shandera | | CHECKED BY A3 | |
| PART FILE 421016T17 | | REV 1 | |

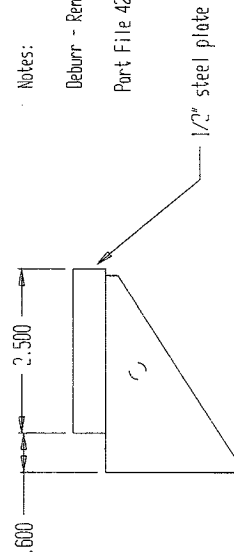
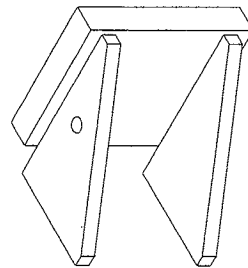


1/2" steel plate
4 required



Part C 421016T06

Part D 421016T06

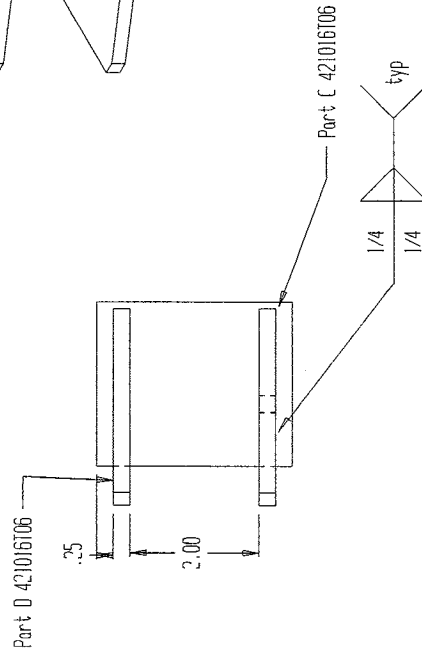
Part A
2 requir

Notes:

Deburr - Remove All Sharp Edges

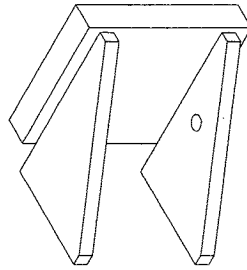
Part File 421016T18



1/2" steel plate

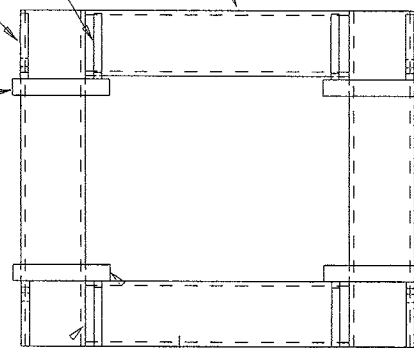


Part D 421016T06



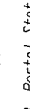
Part C 421016T06



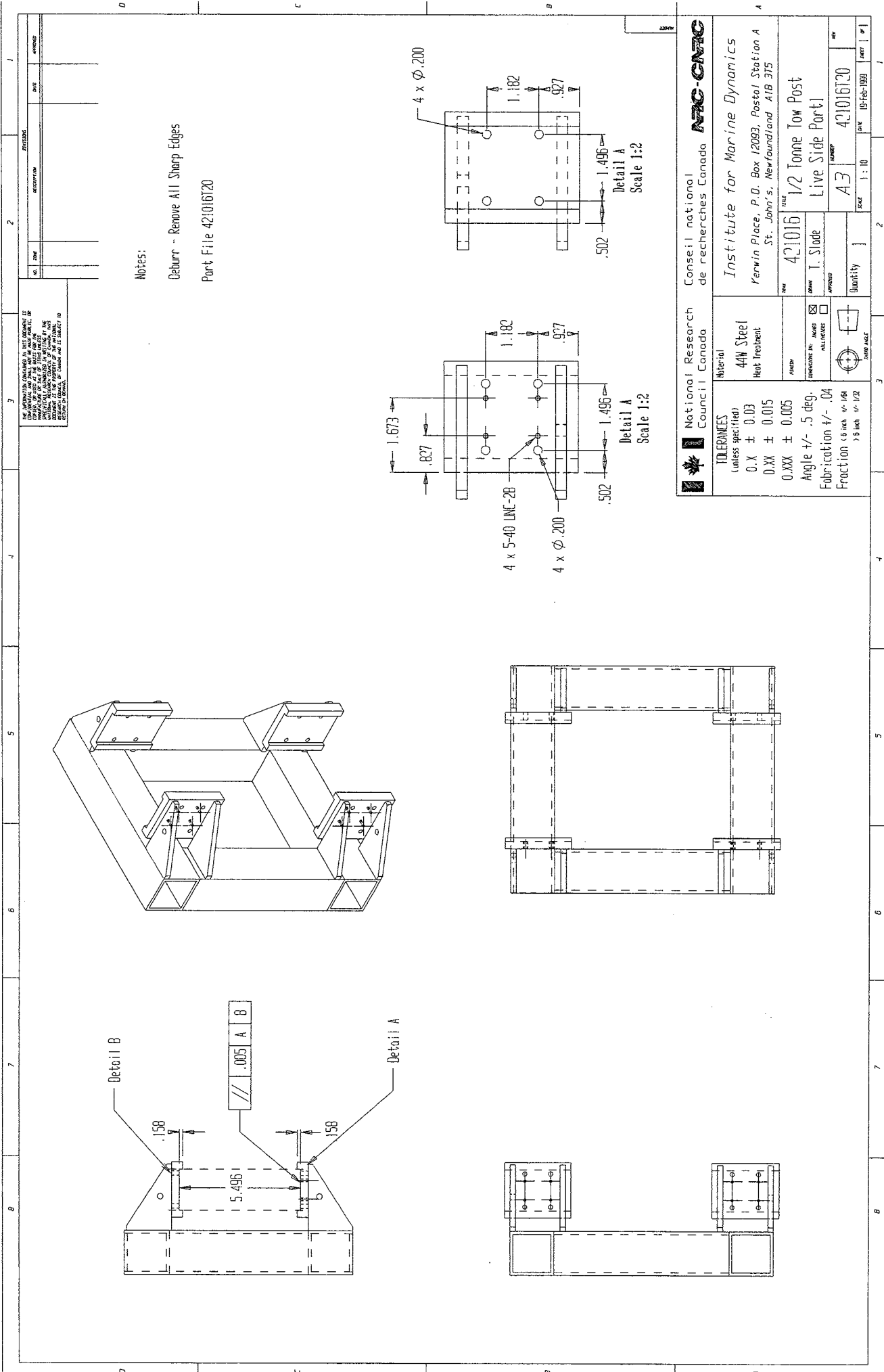
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|---|--|--|--|--|--|------------------------|--|
|  National Research Council Canada | | Institute for Ocean Technology Perwin Place, P.O. Box 12093, Postal Station A St. John's, Newfoundland A1B 3T5 | | Yow Restraining Tow Post Bearing Mounts Fabrication | | 421016119 | |
| TOLERANCES (unless specified) 0.X ± 0.03 0.XX ± 0.015 0.XXX ± 0.005 Angle +/- .5 deg. Fabrication +/- .004 Fraction 1/8 inch 1/16 1/32 | | Material 05 noted Heat Treatment | | MAKE 421016 | | DATE 13-Apr-11-2004 | |
| QUANTITIES 2 | | FINISH SURFACE IN. <input checked="" type="checkbox"/> JAWES MILL-FINISH | | DRAWN R. Shandera | | SCALE 1:2 | |
| DIMENSIONS  1/8 inch DIA | | APPROVED | | REV | | DATE | |



Part A 421016T06 ————— Part A 421016T18

| | | | |
|--|---|--|---|
|  | National Research Council Canada | Consente national de recherches Canada |  |
| TITULARS (unless specified) 0.XX ± 0.03 0.XX ± 0.015 0.XXX ± 0.005 Angle 4/- .5 deg. Fabrication 4/- .04 Fraction 1/8 inch 4/- .004 > 6 inch 4/- .002 | Material 44W Steel Heat Treatment | Institute for Ocean Technology Fernin Place, P.D. Box 12093, Postal Station A St. John's, Newfoundland A1B 3T5 | N7C-CN7C |
| Finish SURFACING BY: <input checked="" type="checkbox"/> JAWNE <input type="checkbox"/> ALLIANCE | Date 421016 | Title Yaw Restraining Tow Post Front Frame Fabrication | Rev 421016119 |
| Quantity 1 Unit  | Drawing R. Shandera | Scale 1 : 10 | Date 19-Feb-003 |

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Notes:
 Deburr - Remove All Sharp Edges
 Part File 421016T20

| | | | |
|---|--|--|--|
| National Research Council Canada Conseil national de recherches Canada | | ATAC-CNRC Institute for Marine Dynamics P.O. Box 12093, Postal Station A St. John's, Newfoundland A1B 3T5 | |
| TOLERANCES Unless specified 0.X ± 0.03 0.XX ± 0.015 0.XXX ± 0.005 Angle +/- .5 deg. Fabrication +/- .04 Fraction 1/8 inch 1/16 | | Material 44W Steel Heat Treatment Finish DIMENSIONS IN INCHES DECIMALS IN INCHES FRACTIONS IN INCHES DECIMALS IN MILLIMETERS FRACTIONS IN MILLIMETERS DECIMALS IN METERS FRACTIONS IN METERS | |
| Part No. 421016 Title 1/2 Tonne Tow Post Live Side Part | | Date 15-Feb-1993 Drawn By T. Slade Checked By A3 Part No. 421016T20 | |
| Quantity 1 | | Scale 1:2 | |

| | | | | | | | | |
|---|---|---|---|---|---|---|---|---|
| 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|---|---|---|---|---|---|---|---|---|

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Notes:

Deburr - Remove All Sharp Edges

Part File 421016121

Part A

3" x 3" x 3/16" box tube

1 required

Part B

2" steel plate

1 required

Part C

1/4" steel

2 required

Dimensions:

40.000

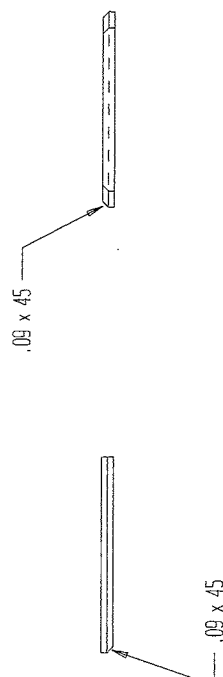
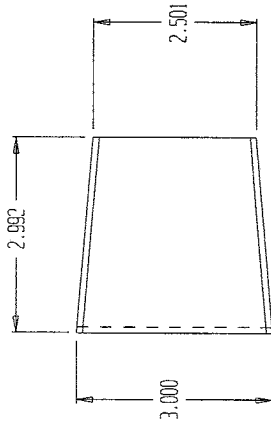
4.500

4.000

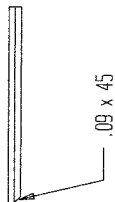
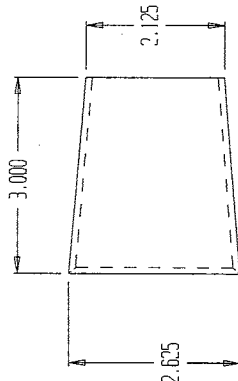
2.550

3.000

| | | | |
|--|--|--|--|
| National Research Council Canada Conseil national de recherches Canada | | NRCC-CNRC Institute for Ocean Technology | |
| 44W Steel Heat Treated | | Yaw Restraining Tow Post Post Pieces | |
| Material 44W Steel Heat Treated | | Yaw Restraining Tow Post Post Pieces | |
| FINISH MILL FINISH | | YAW 421016 | |
| DIMENSIONS IN INCHES MILLIMETERS | | DRAWN R. Stendera | |
| TOLERANCES (unless specified) 0.X ± 0.03 0.XX ± 0.015 0.XXX ± 0.005 Angle 1/4° - .5 deg. Fabrication 1/4° - .04 Fraction 1/8 inch 1/4 1/2 | | APPROVED 421016121 | |
| QUANTITY As Noted | | DATE 08-April-2004 | |



Part A
3/16" steel plate
2 required



Part B
3/16" steel plate
2 required

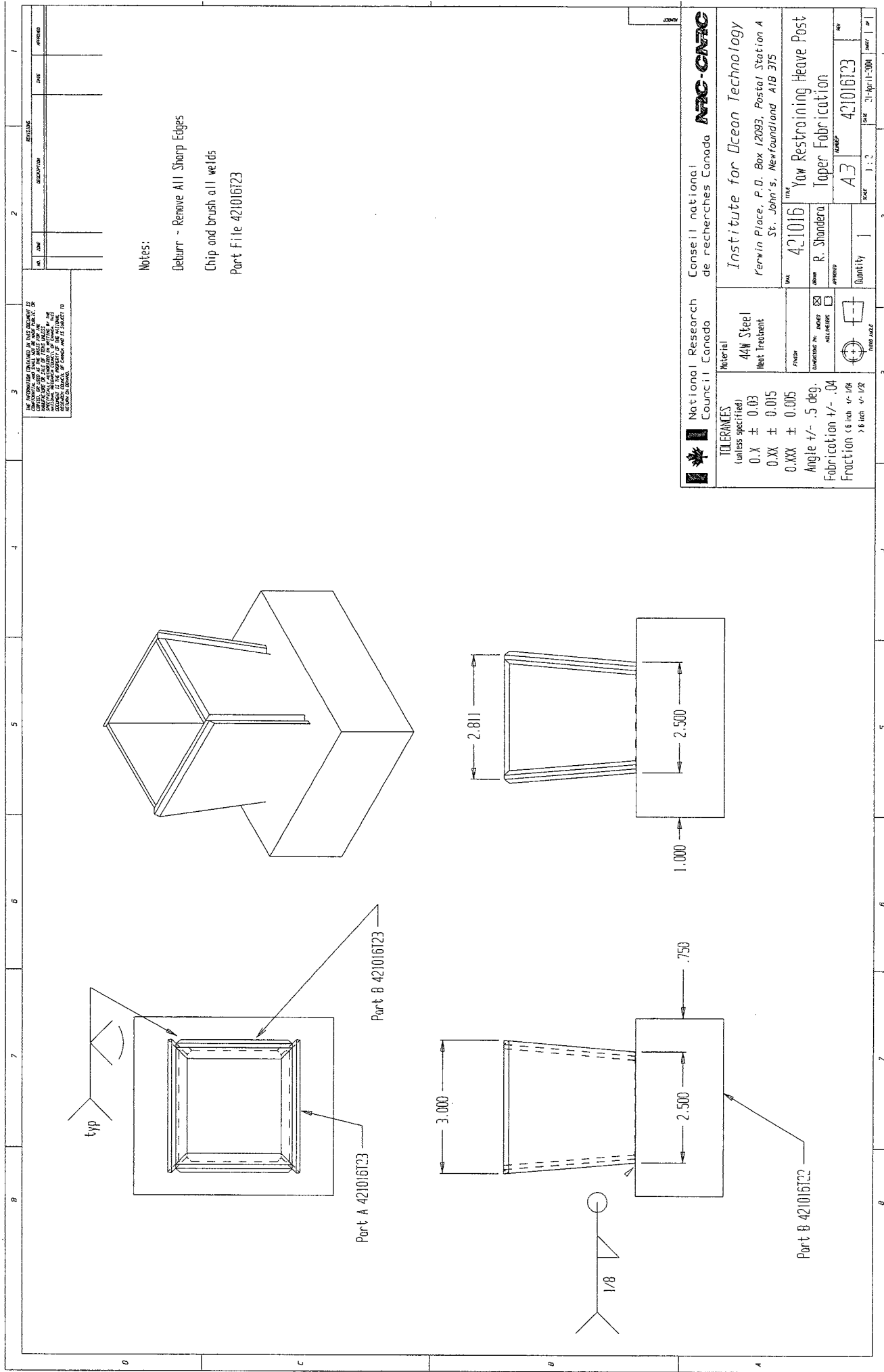
Notes:
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Part File 421016122

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| | | | |
|--|--|--|--|
| | | National Research Council of Canada Conseil national de recherches Canada | |
| TOLERANCES (unless specified) 0.X ± 0.03 0.XX ± 0.015 0.XXX ± 0.005 Angle +/- .5 deg. Fabrication +/- .04 Fraction < 8 inch +/- .004 > 8 inch +/- .012 | | Material As noted Heat treatment | |
| FINISH SHARPENING IN: <input checked="" type="checkbox"/> JACKS <input checked="" type="checkbox"/> ALL INTERIORS <input checked="" type="checkbox"/> | | PART 421016 Yaw Restraining Tow Post Taper Pieces | |
| DRAWN R. Shandera | | DATE 421016 | |
| APPROVED | | REV A3 421016122 | |
| QUANTITY 05 noted | | SCALE 1:2 | |
| SHEET 1 of 1 | | DATE 21-April-2004 | |

Institute for Ocean Technology
 Yerrwin Place, P.O. Box 12093, Postal Station A
 St. John's, Newfoundland A1B 3T5

NTC-CNRC

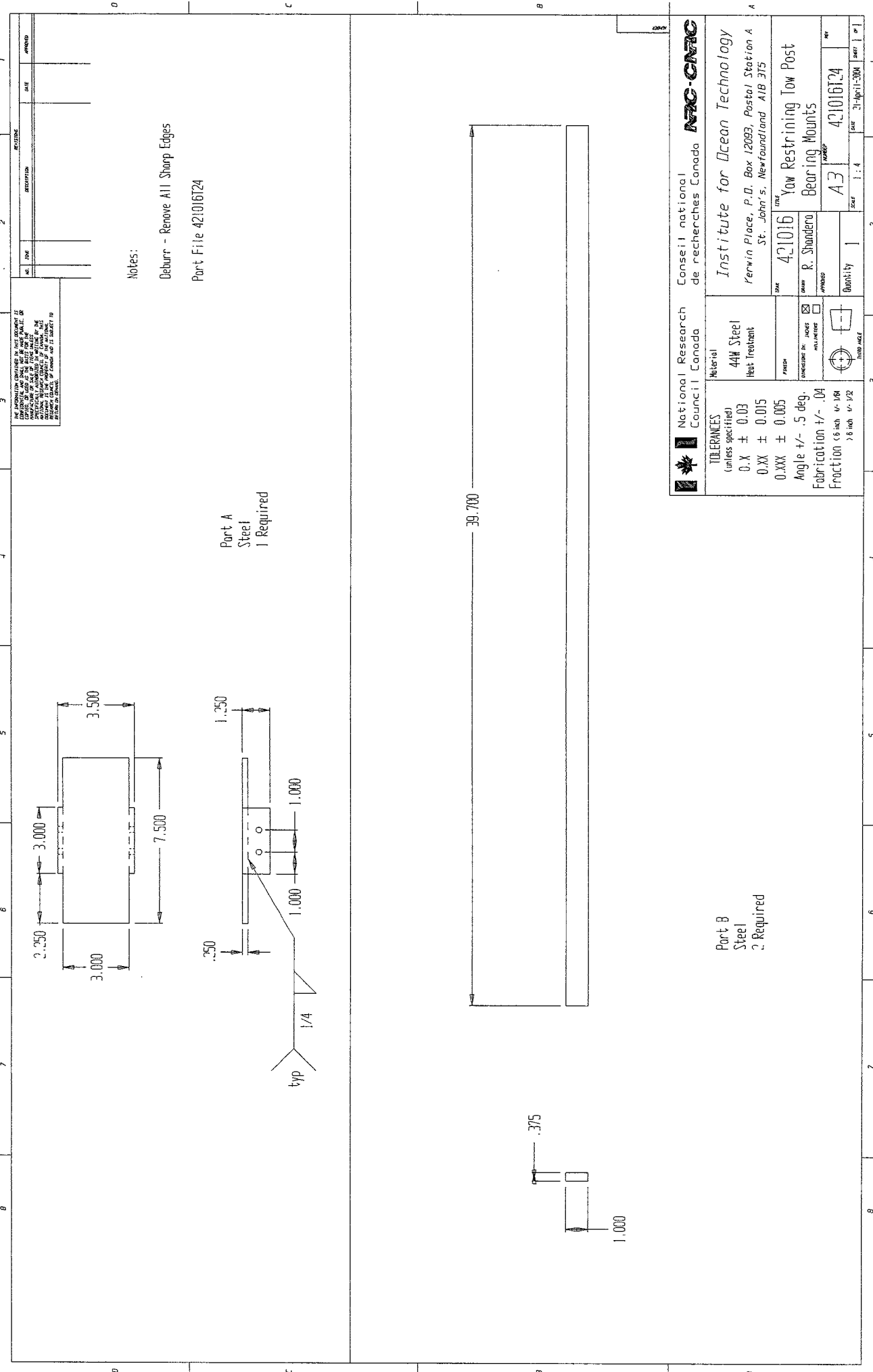


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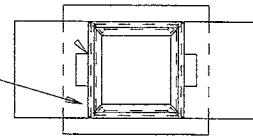
Notes:

- Deburr - Remove All Sharp Edges
- Chip and brush all welds
- Part File 421016123

| | | | |
|---|--|---|--|
| National Research Council Canada Conseil national de recherches Canada | | INAC-CNRIC Institute for Ocean Technology | |
| TOLERANCES (unless specified) 0.X ± 0.03 0.XX ± 0.015 0.XXX ± 0.005 Angle +/- .5 deg. Fabrication +/- .04 Fraction < 6 inch 1/8 > 6 inch 1/2 | | Material 44W Steel Heat Treatment | |
| FINISH QUALITY IN JACKS <input checked="" type="checkbox"/> ALLIGERS <input type="checkbox"/> | | DATE 421016 Yaw Restraining Heave Post Taper Fabrication | |
| DRAWN R. Shandera | | CHECKED A3 | |
| QUANTITY 1 | | SCALE 1:2 | |
| DATE 21-Apr-11-2004 | | DRAWN 1 | |



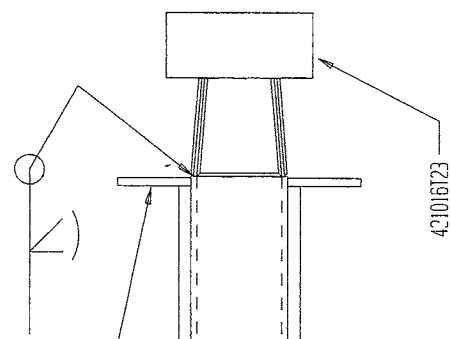
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|--|---|--|--|-------------------|
| | National Research Council Canada Conseil national de recherches Canada | | Institute for Ocean Technology Perwin Place, P.O. Box 12093, Postal Station A St. John's, Newfoundland A1B 3T5 | |
| | Material 44H Steel Heat Treatment | | Part 421016 Yaw Restraining Tow Post Bearing Mounts | |
| TOLERANCES (unless specified) 0.X ± 0.03 0.XX ± 0.015 0.XXX ± 0.005 Angle +/- .5 deg. Fabrication +/- .04 Fraction 1/8 inch 1/16 1/32 | | Part 421016 Yaw Restraining Tow Post Bearing Mounts | | Part 421016124 |
| FINISH ENGINEERING IN INCHES MILLIMETERS | | DRAWN R. Shandera | | APPROVED A3 |
| THIRD ANGLE | | Quantity 1 | | Scale 1 : 4 |
| | | DATE 21-April-2004 | | SHEET 1 OF 1 |



Notes:




Deburr - Remove All Sharp Edges

Part File 421016T25



Part C 421016T21

Part B 421016124

| | | | | | | |
|--|-------------------------------------|--|--|--|---|--|
|  | National Research Council of Canada | | Conseil national de recherches Canada | |  | |
| | | | | | | |
| TOLERANCES (unless specified) 0.X ± 0.03 0.XX ± 0.015 0.XXX ± 0.005 Angle +/- .5 deg. Fabrication +/- .04 Fraction | | | Material 44W Steel Heat treatment | | <i>Institute for Ocean Technology</i> Verwin Place, P.O. Box 12003, Postal Station A St. John's, Newfoundland A1B 3T5 | |
| FINISH BURNISHED BY: <input checked="" type="checkbox"/> HONES <input type="checkbox"/> POLYMERIZED <input type="checkbox"/> | | | NAME 421016 | | Yaw Restraining Tow Post Post Fabrication | |
|  PARTS MADE | | | QUANTITY 1 | | A3 A3 421016125 19-Feb-1995 1:4 1 | |

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REVISIONS

| NO. | DATE | DESCRIPTION |
|-----|------|-------------|
| 1 | | |

Notes:

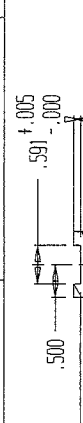
Deburr - Remove All Sharp Edges

Part File 421016126

SECTION A-A
SCALE 1:2

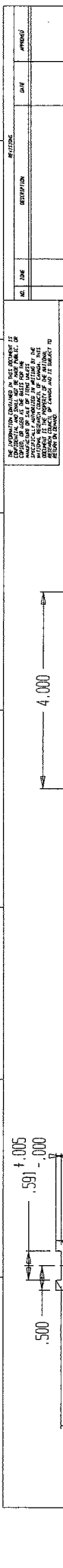


SECTION B-B
SCALE 1:2



16 x 2.361 = 37.776

34 x 5-40-UNC-3B



| | |
|------------------------------------|---------------------|
| TOLERANCES (unless specified) | 0.X ± 0.03 |
| | 0.XX ± 0.015 |
| 0.XXX ± 0.005 | Angle +/- .5 deg. |
| | Fabrication +/- .04 |
| Fraction 1/8 1/4 3/8 1/2 5/8 3/4 1 | |

| | | |
|-----------------|--|--------------------------|
| Material | | 44W Steel |
| Heat treatment | | Heat treatment |
| FINISH | | |
| SURF. TREATMENT | | |
| DRAWN | | R. Shandera |
| APPROVED | | |
| PART NO. | | 421016 |
| TITLE | | Yaw Restraining Tow Post |
| POST MACHINING | | |
| QUANTITY | | 1 |
| SCALE | | 1:2 |
| DATE | | 21 April 2004 |
| REV | | |

NATIONAL RESEARCH COUNCIL OF CANADA

Conseil national de recherches Canada

NTC-CRTC

Institute for Ocean Technology

Yerwin Place, P.O. Box 12093, Postal Station A

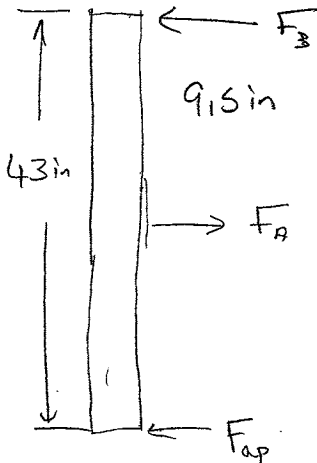
St. John's, Newfoundland A1B 3T5

8.0 Appendix B



Références / Références

Titre / Titre

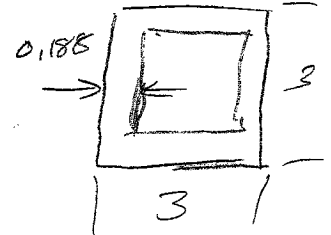


$$F = ma$$

$$m = 2000 \text{ kg}$$

$$a = 3 \text{ m/s}^2$$

$$\begin{aligned} F &= (2000 \text{ kg})(3 \text{ m/s}^2) \\ &= 6 \text{ kN} \\ &= 1348.9 \text{ lbs} \end{aligned}$$



$$\sum M_B = 0 = -F_{AP}(43) + F_A(9.5)$$

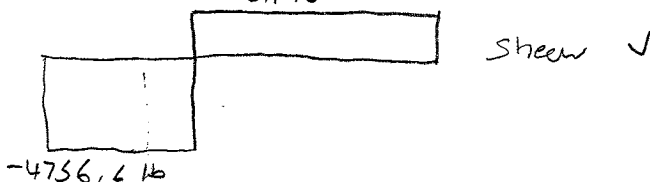
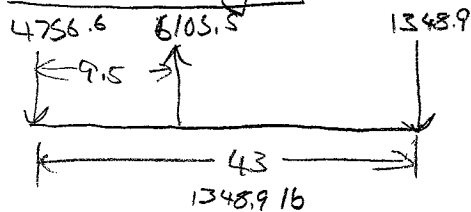
$$\begin{aligned} F_A &= \frac{F_{AP}(43)}{9.5} \\ &= \frac{(1348.9 \text{ lb})(43)}{9.5} \end{aligned}$$

$$= 6105.5 \text{ lb} = 27.2 \text{ kN (Bearing OK)}$$

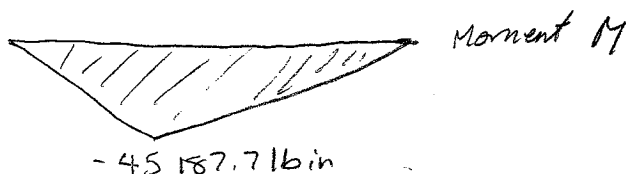
$$\sum F_x = 0 = -F_A + F_B - 1348.9 \text{ lb}$$

$$\begin{aligned} F_A &= 6105.5 \text{ lb} - 1348.9 \text{ lb} \\ &= 4756.6 \text{ lb} = 21.2 \text{ kN (Bearing OK)} \end{aligned}$$

Beam Strength



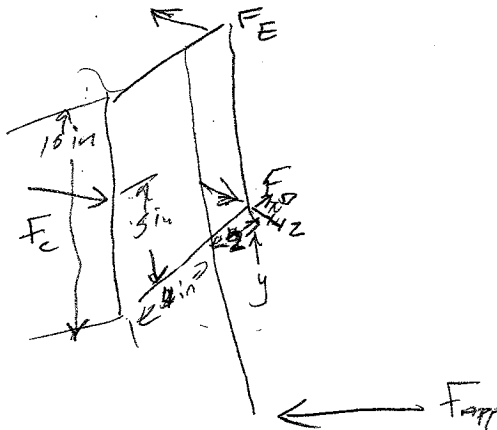
$$\begin{aligned} \sigma &= \frac{MC}{I} \\ &= \frac{(45187.7 \text{ lb in})(1.5 \text{ in})}{\frac{1}{12}(3^4 - 2.624^4)} \\ &= 24213.8 \text{ PSI} \\ \Rightarrow FS &= \frac{58 \times 10^3}{24 \times 10^3} \\ &= 2.4 \end{aligned}$$





References / Références

Title / Titre



$$\sum M_y = 0 = F_C(6) - F_{APP}(4)$$

$$\begin{aligned} F_C &= \frac{2}{3} F_{APP} \\ &= 2(1348.916) \\ &= 449.616 \end{aligned}$$

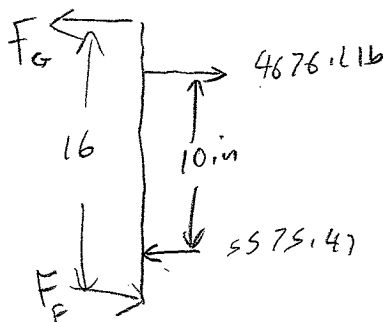
$$\sum M_x = 0 = -F_E(10) + 449.6(5) + 1348.9(33.16)$$

$$\begin{aligned} F_E &= \frac{(449.6)(5) + 1348.9(33.16)}{10} \\ &= 4676.216 \end{aligned}$$

$$\sum F_z = 0 = F_D - F_E + F_C - 1348.9$$

$$\begin{aligned} F_D &= F_E + 1348.9 - F_C \\ &= 4676.17 + 1348.9 - 449.6 \\ &= 5575.47 \end{aligned}$$

Pivot Post





Date _____

Title / Titre

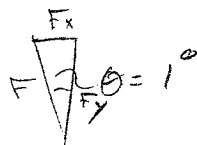
$$FS = \frac{58 \times 10^3}{41 \times 10^{-3}} = 1.4$$



Références / Références

Titre / Titre

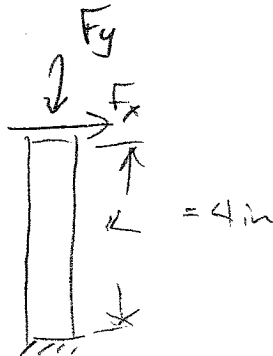
treaded rod



$$F = 500 \text{ lbs}$$

$$F \sin \theta = F_x$$

$$F_x = 8.7 \text{ lb.}$$



$$F_y = F \cos \theta$$

$$= 499.9$$

$$\sigma = \frac{P}{A} + \frac{M C}{I}$$

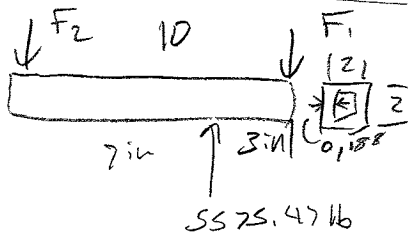
$$= \frac{499.9 \text{ lb}}{\pi (0.75)^2} + \frac{(8.7)(4 \text{ in})(0.25 \text{ in})}{\frac{\pi}{4} (0.75)^4}$$

$$= 5377 \text{ lb OK.}$$

Deflection of Cantilever

$$\delta = \frac{P L^3}{3 E I} = \frac{(1500 \text{ lb})(33 \text{ in})^3}{3 (29 \times 10^6 \text{ psi}) (\frac{1}{12} (3^4 - 2.624^4))}$$

$$= 0.25 \text{ in of deflect at max load \& length.}$$



$$\sum M_1 = 0 = (-5575.47 \text{ lb})(3 \text{ in}) + F_2 (10 \text{ in})$$

$$F_2 = 1672.6 \text{ lb}$$

$$F_1 = (5575.47 - 1672.6) \text{ lb}$$

$$= 3902.8$$

$$M = 11708.4 = 3902.8 (3)$$

$$\sigma = \frac{M C}{I} = \frac{(11708.4 \text{ lb in})(1)}{\frac{1}{12} (2^4 - 1.624^4)} = 15.5 \text{ ksi}$$

$$\text{OK}$$

Twinall &
Marshall
(P 186)



NRC · CNRC

Project / Projet heavy post

Page 5 of / de

Name / Nom Robert Shandera

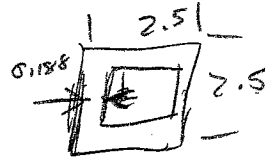
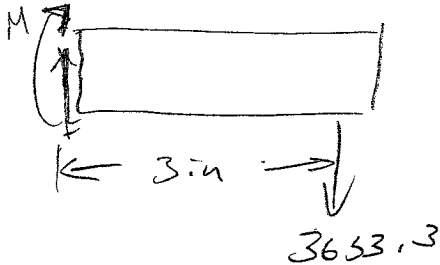
Date March 2004

Approved by / Approuvé par

Date

Références / Références

Titre / Titre



$$M = -(3653.3)(3) \\ = -10959.9 \text{ lb}\cdot\text{in}$$

$$\sigma = \frac{Mc}{I}$$

$$= \frac{(10959.9)(1.75 \text{ in})}{\frac{1}{12}(2.5^4 - 2.124^4)}$$

$$= 8786 \text{ PSI} < 58000 \text{ PSI OK}$$